# NATIONAL UNIVERSITY OF ENGINEERING COLLEGE OF CHEMICAL AND TEXTILE ENGINEERING <br> Professional School of Chemical Engineering 

## PLANT DESIGN

PI 525 A


Design of an acetone production plant via catalytic dehydrogenation of isopropyl alcohol

## Professors:

- Eng. Rafael Chero Rivas
- Eng. Víctor León Choy

Work Team:

- CONDORI LLACTA, ALEX RENZO
- FLORES ESTRADA, FABRIZIO ALEXANDER
- FLORES GIL, KEVIN ANDREI
- SOTO MORENO, MIGUEL EDUARDO

20114099K
20114049C
20112140C
20114003C

## Content

1. Introduction ..... 1
Context ..... 1
2. Objectives ..... 2
General objective: ..... 2
Specific objectives: ..... 2
3. REVIEW OF THE TECHNOLOGY APPLIED TO THE CASE STUDY ..... 3
Base Process Block Diagram. ..... 3
Base Case Process Flow Diagram ..... 5
4. BALANCE OF SUBJECT MATTERS ..... 7
5. DETERMINATION OF FIXED CAPITAL ..... 8
6. DETERMINATION OF THE BALANCE PRICE ..... 11
7. BIBLIOGRAPHY ..... 16
8. Annexes ..... 17

## DESIGN OF AN ACETONE PRODUCTION PLANT VIA CATHYTIC <br> DEHYDROGENATION OF ISOPROPYL ALCOHOL

## 1. INTRODUCTION

In Process Engineering, a good design of the plant determines the future of the project in which it is intervened; and the high national and international competition for the market has the effectiveness of finding solutions to problems and nurturing a policy of taking initiatives, as well as incorporating innovation in technologies and processes. Therefore having a clear idea of methodology, creativity and knowledge is essential to guarantee what you want to get as a product or service. The present report shows the development of the design of a plant for the production of Acetone from Isopropyl Alcohol (IPA) by catalytic dehydrogenation which is intended to be adjustable to a capacity of 20000 tonnes per year, optimum, controllable, with Low levels of pollutants and low production costs.

## Context

Acetone is widely used in industry, in the manufacture of Methyl Methacrylate (MMA), Methacrylic Acid, Methacrylates, Bisphenol A, among others, but it can also be used as:

- Solvent for most plastics and synthetic fibers.
- Ideal for thinning fiberglass resins.
- Cleaning of wool and fur garments.
- Clean fiberglass tools and dissolve epoxy resins.
- Cleaning of microcircuits and electronic parts.
- It is used as a volatile component in some paints and varnishes.
- It is useful in the preparation of metals before painting.
- Acetone is often used as a nail cleaner.

In the production of Acetone different methods are presented, of which three are outstanding: Cumene process, oxidation process of polypropylene and the process of dehydrogenation of Isopropyl alcohol. The Cumene process is the most common worldwide, but as a byproduct, is benzene (carcinogen) lowering the purity of Acetone and increasing production costs by separation. The oxidation of polypropylene has a low conversion of acetone and the purity of the reactants should be $99 \%$. In the dehydrogenation of IPA, high purity Acetone is obtained, the IPA can be used in aqueous solution, and the conversion of acetone is high and has no substances, which are significantly hazardous to health. This process leaves us as the main product acetone from which its multiple uses were mentioned, and as secondary products: Hydrogen, widely used in the chemical industry: ammonia synthesis, refinery processes, coal treatment, among others. In this report, we opt for the dehydrogenation process of IPA, which offers great advantages and results with low production costs.

## 2. OBJECTIVES

## General objective:

- Carry out the conceptual design of a $20,000 \mathrm{t}$ / year production plant of $99.9 \%$ molar acetone by catalytic dehydrogenation of isopropyl alcohol and check its economic viability by finding the equilibrium price for the requested acetone production capacity.


## Specific objectives:

- To structure the preliminary design of an acetone production plant, so that a careful and correct choice of the required operating equipment is made, specifying
the dimensions, materials, costs and operating capacity of each of them, in the proper sequence.
- To look for the conditions of optimization of the productive process of the acetone from isopropyl alcohol through the implementation of recycle streams, energy integration and proper use of each equipment involved.


## 3. REVIEW OF THE TECHNOLOGY APPLIED TO THE CASE STUDY

There are different methods of obtaining acetone, among which one of the most used in the pharmaceutical industry is the process of catalytic dehydrogenation of isopropyl alcohol due to the high purity of the product; this process consists of a set of operations well known and used by process engineers.

## Base Process Block Diagram

The base case study deals with the production of acetone from isopropyl alcohol by the dehydrogenation reaction in the gas phase of the isopropyl alcohol in the presence of the catalyst, then undergoes cooling, condensation and purification (separation of impurities from the product - IPA, water) To obtain acetone with high purity (99.9\%), with the said block diagram of the base process would be as follows:


In the block diagram shown, the main operations that make up the process are presented, which are justified by the following reasons:

- MIXING: The concentration of IPA in the feed is different from that recirculated from the top of the IPA distillation column, which contains small quantities of the product (acetone) that could not be separated in the acetone distillation column. For this reason, it is necessary to mix these two streams before entering the vaporizer, in order to homogenize the properties of the mixture.

The resulting mixture of IPA, water and a depreciable amount of acetone requires to be vaporized, since the catalytic dehydrogenation reaction is carried out in the gas phase in order to improve the contact between the catalyst and the reactant mixture.

- REACTOR: In order to carry out the reaction it is necessary to bring the feed mixture into contact with the catalyst, to supply the heat of reaction because it is endothermic, to feed the reactants and to withdraw the products in such a way as to favor the kinetics And selectivity. All these operations must be carried out in a reactor that allows the reactants to be converted into desired products.
- CONDENSER AND COOLING: The reactor outlet current is at high temperatures, for this reason it is necessary to cool this stream, to condense the IPA, acetone and water vapors in order to improve the separation of the liquid and gaseous phases in the Flash tab.

The reactor output stream is made up of product (acetone), water, unreacted isopropyl alcohol and hydrogen. In order to separate the desired product from the other components it is necessary to carry out several separation operations, which are:

- PHASE SEPARATOR: The formation of a liquid and gaseous phase after cooling allows using a flash separation, making it possible to separate all the hydrogen from the liquid phase because it has low (negligible) solubility. In the gas phase hydrogen and vapors of IPA, acetone and water will be in equilibrium with their liquids.
- ACETONE DISTILLATION: The liquid phase separator is fed to the distillation column, which is made up of water, IPA, acetone and traces of hydrogen. The separation in this operation is carried out taking advantage of the difference of volatilities of the 3 components, this allows to obtain the product (acetone) to the desired specification (99.9\%) with a small amount of IPA at the top as the lighter components And a bottom stream consisting of IPA, water and a negligible amount of acetone to be separated in the next step.
- DISTILLATION OF IPA: The feed of this column is formed by the bottoms of the acetone distillation column. This operation allows obtaining a current of IPA and acetone (negligible) by the stop to be recirculated to the process and a bottom stream formed by IPA and water, whose concentration is limited by the capacity of the wastewater treatment system.


## Base Case Process Flow Diagram

## PFD: Acetona vía deshidrogenación catalítica del alcohol isopropílico (IPA)



## 4. BALANCE OF SUBJECT MATTERS

Table: CURRENT MATERIAL BALANCE

| No. CURRENT | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 | 13 | 14 | 15 | 16 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Mass flow (kg / h) | 3280.27 | 3772.00 | 3772.00 | 3772.00 | 3772.00 | 3772.00 | 3772.00 | 3772.00 | 424.06 | 3347.94 | 2450.98 | 896.90 | 491.74 | 405.16 | 0.06 | 424.12 |
| Molar flow (kmol / hr) | 70.7809 | 81.5159 | 81.5159 | 81.5159 | 81.5159 | 128.9453 | 128.9453 | 128.9453 | 53.4962 | 75.4490 | 42.1986 | 33.2199 | 10.7350 | 22.485 | 0.0305 | 53.527 |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 25.0 | 33.5 | 33.5 | 117.0 | 180.00 | 387.65 | 92.2 | 32.0 | 32.0 | 32.0 | 45.8 | 107.0 | 96.45 | 130.0 | 45.8 | 38.9 |
| Pressure (kPa) | 99.8 | 99.8 | 400.0 | 380.0 | 360.0 | 340.0 | 320.0 | 300.0 | 300.0 | 300.0 | 280.0 | 300.0 | 250.0 | 260.0 | 99.8 | 99.8 |
| Components (kmol / <br> h) |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| Hydrogen | 0.0000 | 0.0000 | 0.0000 | 0.0000 | 0.0000 | 47.4294 | 47.4294 | 47.4294 | 47.3989 | 0.0305 | 0.0000 | 0.0000 | 0.0000 | 0.0000 | 0.0305 | 47.4294 |
| Acetone | 0.0000 | 2.1429 | 2.1429 | 2.1429 | 2.1429 | 49.5722 | 49.5722 | 49.5722 | 5.2729 | 44.2993 | 42.1564 | 2.1429 | 2.1429 | 0.0000 | 0.0000 | 5.2729 |
| Isopropyl alcohol | 47.6497 | 52.6993 | 52.6993 | 52.6993 | 52.6993 | 5.2699 | 5.2699 | 5.2699 | 0.1759 | 5.0940 | 0.0422 | 5.0518 | 5.0496 | 0.0022 | 0.0000 | 0.1759 |
| Water | 23.1312 | 26.6737 | 26.6737 | 26.6737 | 26.6737 | 26.6737 | 26.6737 | 26.6737 | 0.6485 | 26.0252 | 0.0000 | 26.0252 | 3.5426 | 22.4826 | 0.0000 | 0.6485 |

## 5. DETERMINATION OF FIXED CAPITAL

- For the determination of Fixed Capital Investment, it is necessary to make a good estimate of total equipment costs (Ce), since the other components of fixed capital (facilities, pipelines, services, engineering, etc.) are determined on a To this first calculation.
- To determine the equipment costs (Ce) the following equation was used:

$$
\mathbf{C e}=\mathbf{a}+\mathbf{b S}^{\mathbf{n}}
$$

Where:
S: Size of the parameter used to determine the cost of the equipment
A, b and n: Constants obtained from Table 6.6 of the book "Chemical Engineering Design

- Gavin Towler"
- The costs obtained in this way are based on equipment of the Gulf Coast (US) of the year 2006, for this reason the Chemical Engineering (IC) Cost Indices were used to obtain the cost of the equipment in the present year (2015).

$$
\mathrm{Ce}_{2015}=\mathrm{Ce}_{2006} \times \frac{\mathrm{IC}_{2015}}{\mathrm{IC}_{2006}}
$$

| Cost Index |  |
| :---: | :---: |
| Year | IC |
| 2006 | 499.6 |
| 2015 | 573.1 |

Finally, a correction factor for location (FL) is applied to obtain the costs in Peru and a correction factor for material (FM), since the calculated equipment costs are based on carbon steel ( $\mathrm{FM}=1$ ) and it is Necessary to determine the costs of the equipment in stainless steel 304 ( $\mathrm{FM}=1.3$ ).

Table 6.5. Materials Cost Factors, $f_{m}$, Relative to Plain Carbon Steel

| Material | $\boldsymbol{f}_{\boldsymbol{m}}$ |
| :--- | :--- |
| Carbon steel | 1.0 |
| Aluminum and bronze | 1.07 |
| Cast steel | 1.1 |
| 304 stainless steel | 1.3 |
| 316 stainless steel | 1.3 |
| 321 stainless steel | 1.5 |
| Hastelloy C | 1.55 |
| Monel | 1.65 |
| Nickel and Inconel | 1.7 |

Table 6.7. Location Factors

| Country | Region | Location Factor |
| :--- | :--- | :---: |
| United States | Gulf Coast | 1.00 |
|  | East Coast | 1.04 |
|  | West Coast | 1.07 |
|  | Midwest | 1.02 |
| Canada | Ontario | 1.00 |
|  | Fort McMurray | 1.60 |
| Mexico |  | 1.03 |
| Brazil | imported | 1.14 |
| China | indigenous | 1.12 |
|  |  | 0.61 |
| Japan |  | 1.26 |
| SE Asia | 1.12 |  |
| Australia | 1.21 |  |
| India | 1.02 |  |
| Middle East |  | 1.07 |
| France | 1.13 |  |
| Germany |  | 1.11 |
| Italy | 1.14 |  |
| Netherlands |  | 1.19 |
| Russia | 1.53 |  |
| United Kingdom |  | 1.02 |


| COSTO DEEQUIPOS (Ce) ESCALADO Y LOCALIZADO CON CORRECCION DE MATERIAL (\$ del año 2015) |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Nombre del Equipo |  |  | $\mathbf{C e}=\mathrm{a}+\mathrm{bS} \wedge$ n |  |  |  |  | $\begin{gathered} \text { Ce (\$ del } \\ \text { año } 2006 \text { - } \\ \text { U.S. Gulf } \\ \text { Coast) } \end{gathered}$ | IC | FL | FM = 1.3 (ss 304) |
| Codigo |  |  | $\begin{gathered} \text { Cantida } \\ d \end{gathered}$ | Tamaño del Equipo (S) | Unidad es | a | b | n |  | Ce escalado (\$ del año 2015 - <br> U.S. Gulf Coast) | Ce escalado y localizado (\$ del año 2015 - Peru) | Ce escalado y localizado con correcion de material (\$ del año 2015 - Peru) |
|  | Tanque de techo conico (almac IPA) |  | 5 | 393 | m3 | 5700 | 700 | 0.7 | 257,539 | 295,428 | 326,430 | 424,358 |
| M-110 | Mezclador de IPA | T anque de mezclado | 1 | 1.9 | m3 | 5700 | 700 | 0.7 | 6,810 | 13,585 | 15,011 | 19,514 |
|  |  | Mezclador de helice | 1 | 0.30 | kW | 4300 | 1920 | 0.8 | 5,033 |  |  |  |
| L-111 | Bomba alimentac IPA | Bomba centrifuga | 2 | 1.35 | L/s | 3300 | 48 | 1.2 | 6,738 | 10,571 | 11,680 | 15,184 |
|  |  | Motor a prueba de explosion | 2 | 0.41 | kW | 920 | 600 | 0.7 | 2,477 |  |  |  |
| V-112 | Vaporizador de IPA |  | 1 | 48 | m2 | 11000 | 115 | 1 | 16,572 | 19,010 | 21,005 | 27,306 |
| E-113 | Pre-calentador de IPA |  | 1 | 9 | m2 | 11000 | 115 | 1 | 12,073 | 13,849 | 15,302 | 19,893 |
| R-120 | Reactor tubular |  | 1 | 238 | m2 | 11000 | 115 | 1 | 38,411 | 44,062 | 48,686 | 63,292 |
| Q-122 | Horno Cilindrico |  | 1 | 1.39 | MW | 53000 | 69000 | 0.8 | 142,701 | 163,695 | 180,873 | 235,135 |
| E-130 | Enfriador |  | 1 | 15 | m2 | 11000 | 115 | 1 | 12,691 | 14,557 | 16,085 | 20,911 |
| H-132 | Separador de fases |  | 1 | 459 | kg | -10000 | 600 | 0.6 | 10,559 | 12,112 | 13,384 | 17,399 |
| D-140 | Destilador <br> Acetona | Recipiente a presion | 1 | 3775 | kg | -10000 | 600 | 0.6 | 56,926 | 84,429 | 93,289 | 121,276 |
|  |  | Platos perforados | 58 | 1.25 | m | 100 | 120 | 2 | 16,675 |  |  |  |
| L-143 | Bomba reflujo Acetona | Bomba centrifuga | 2 | 2.50 | L/s | 3300 | 48 | 1.2 | 6,888 | 10,181 | 11,250 | 14,624 |
|  |  | Motor a prueba de explosion | 2 | 0.05 | kW | 920 | 600 | 0.7 | 1,987 |  |  |  |
| E-144 | Rehervidor Acetona |  | 1 | 26 | m2 | 11000 | 115 | 1 | 13,958 | 16,011 | 17,691 | 22,999 |
| D-150 | Destilador IPA | Recipiente a presion | 1 | 520 | kg | -10000 | 600 | 0.6 | 11,978 | 18,810 | 20,784 | 27,020 |
|  |  | Platos perforados | 34 | 0.50 | m | 100 | 120 | 2 | 4,420 |  |  |  |
| E-151 | Condensador IPA |  | 1 | 37 | m2 | 11000 | 115 | 1 | 15,198 | 17,433 | 19,263 | 25,042 |
| L-153 | Bomba reflujo IPA | Bomba centrifuga | 2 | 0.04 | L/s | 3300 | 48 | 1.2 | 6,602 | 9,693 | 10,711 | 13,924 |
|  |  | Motor a prueba de explosion | 2 | 0.001 | kW | 920 | 600 | 0.7 | 1,848 |  |  |  |
| E-154 | Rehervidor IPA |  | 1 | 26 | m2 | 11000 | 115 | 1 | 13,956 | 16,009 | 17,689 | 22,995 |
|  | Tanque de techo conico (almac Acetona) |  | 4 | 393 | m3 | 5700 | 700 | 0.7 | 206,031 | 236,342 | 261,144 | 339,487 |
| Costo Total de Equipos |  |  |  |  |  |  |  |  |  |  |  | 1,430,358 |

- From the costs of major equipment (Ce) we proceed to determine the other components of the Investment in Fixed Capital, for this we use a percentage within the range recommended in Table 4 of Chapter 6 of the book "Plant Design and Economics For Chemical Engineers - Max S. Peters "

| COST OF FIXED CAPITAL (\$ 2015) |  |  |  |
| ---: | :---: | :---: | :---: |
| Item | Range (\%) | Value (\%) | Cost |
| Direct Costs (Fixed Assets) |  |  |  |
| Equipment Cost | $15-40$ | 23 | $1,430,358$ |
| Installation | $6-14$ | 10 | 621,895 |
| Control and Instrumentation | $2-8$ | 3 | 186,568 |
| Pipes and fittings | $3-20$ | 8 | 497,516 |
| Electrical equipment and supplies | $2-10$ | 3 | 186,568 |
| Buildings and structures | $3-18$ | 6 | 373,137 |
| Delimitations | $2-5$ | 3 | 186,568 |
| Services and Facilities | $8-20$ | 13 | 808,463 |
| Ground | $1-2$ | 1 | 62,189 |
| Total Direct Costs |  | $\mathbf{7 0}$ | $\mathbf{4 , 3 5 3 , 2 6 3}$ |
| Indirect Costs (Intangibles) |  |  |  |
| Engineering and supervision | $4-21$ | 8 | 497,516 |
| Construction expenses | $4-16$ | 9 | 559,705 |
| Utility of the contractor | $2-6$ | 4 | 248,758 |
| Contingencies | $5-15$ | 9 | 559,705 |
| Total Indirect Costs |  | $\mathbf{3 0}$ | $\mathbf{1 , 8 6 5 , 6 8 4}$ |
| Total Fixed Capital |  | $\mathbf{1 0 0}$ | $\mathbf{6 , 2 1 8 , 9 4 7}$ |

- We are assuming that the land will be purchased at the beginning of the project, for this reason the investment of this component is recovered at the end of the project as a salvage value (Vs), since the land does not depreciate.


## 6. DETERMINATION OF THE BALANCE PRICE

- Of the components of the total cost of the product, some of them could be determined using production data, material balance, raw material price, investment, labor, etc. From Table 27 of Chapter 6 of the book "Plant Design and Economics for Chemical Engineers - Max S. Peters" you get:

|  | TOTAL PRODUCT COS |  |
| :---: | :---: | :---: |
|  | Components | Range (\%) |
| PRODUCTION OR MANUFACTURING COSTS | Direct Costs <br> Raw Material (IPA) <br> Direct Labor (MO) <br> Supervision <br> Laboratory <br> Maintenance and repairs Services (steam, water, gas, etc.) <br> Patents <br> Total Direct Costs | $\begin{gathered} (10-25) \% \text { MO } \\ (10-20) \% \text { MO } \\ (2-10) \% \text { I } \\ (10-20) \% \text { C } \\ (0-6) \% \text { C } \end{gathered}$ |
|  | Fixed Charges <br> depreciation <br> Asset tax Insurance Total Fixed Charges | $\begin{gathered} (1-4) \% \text { I } \\ (0.4-1) \% \text { I } \end{gathered}$ |
|  | Superintendent's Charges | (5-15)\% C |
| GENERAL EXPENSES | Administrative expenses Sales and Distribution Expenses | $\begin{gathered} 15 \% \mathrm{MO} \\ (2-20) \% \mathrm{C} \\ \hline \end{gathered}$ |


| Nominal Capacity (MTM / Year) | 21.5 |
| :---: | :---: |
| Service factor (\%) | $93.2 \%$ |
| TM IPA / TM Acetone | 1.34 |


| Production Program (MTM / Year) |  |
| :---: | :---: |
| Production Acetone | 20.0 |
| IPA Requirement | 26.7 |

- The IPA price was determined from the CIF value of the import report divided by the net import weight for the United States, accounting for $95 \%$ of the imports of this product.

Reporte de Importaciones por Subpartida Nacional/Pais Origen 2015


For the calculation of some of the components of the cost of the product, we divide them into 3 groups: costs proportional to the level of production, costs proportional to the investment in fixed capital (I) and costs proportional to labor (MO).

| PROPORTIONAL COSTS TO THE LEVEL OF PRODUCTION |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Components | Annual requirement |  | Unit Cost |  | Total cost |
| Raw Material (IPA) | 26767 | TM/year | 1168 | $\$ / \mathrm{TM}$ | $\mathbf{3 1 , 2 7 2 , 7 4 5}$ |

- We determine the direct labor requirement from the following graph, for which we take the line B (average conditions).


FIGURE 6-8
Operating labor requirements for chemical process industries.

| DIRECT WORK <br> HAND |  |
| :---: | :---: |
| H-h / DO.shift | 45 |
| Shifts | 3 |
| DO / year | 340 |
| H-H / year | 45900 |
| \$ / H-H | 3.0 |

PROPORTIONAL COSTS TO THE LABOR

| Components | Range (\%) | Value (\%) | Total Cost |
| :---: | :---: | :---: | :---: |
| Direct Labor (MO) | MO = h-H/year x \$/h-H |  | 137,700 |
| Supervision | $(10-25) \%$ MO | $20 \%$ | 27,540 |
| Laboratory | $(10-20) \%$ MO | $10 \%$ | 13,770 |
| Administrative expenses | $15 \%$ MO | $15 \%$ | 20,655 |
| Total (\$/Year) |  |  | $\mathbf{1 9 9 , 6 6 5}$ |

- As we know the investment in fixed capital ( $\mathrm{I}=\mathrm{Vo}$ ), we can determine the depreciation (D) assuming a 10-year project duration (N), with which the Net Funds Flow (FNF) will subsequently be determined for the entire planning horizon. As mentioned above the land is not depreciated, so it is recovered at the end of the project as a salvage value (Vs).

| PROPORTIONAL COSTS TO THE LABOR |  |  |  |
| :---: | :---: | :---: | :---: |
| Components | Components | Components | Components |
| Direct Labor (MO) | $\mathrm{MO}=\mathrm{h}-\mathrm{H} / \mathrm{año} \times \$ / \mathrm{h}-\mathrm{H}$ |  | 137,700 |
| Supervision | $(10-25) \% \mathrm{MO}$ | $20 \%$ | 27,540 |
| Laboratory | $(10-20) \% \mathrm{MO}$ | $10 \%$ | 13,770 |
| Administrative expenses | $15 \% \mathrm{MO}$ | $15 \%$ | 20,655 |
| Total (\$/Year) |  |  | $\mathbf{1 9 9 , 6 6 5}$ |

- Finally, we determine the other components whose cost is proportional to the cost of the product (C), with which we would have an equation with a single unknown. Solving the equation we would have the following results:

- To determine the price of acetone at break-even point (zero utilities), the total cost of the product would have to be equated with the sales revenue, so the price of acetone will be the total cost of the product between productions of acetone.

> | Acetone Balance Price (ThUS \$ / TM) | 2.13 |
| :--- | :--- |

## DETERMINATION OF WORKING CAPITAL

- We chose a period of 14 days of inventory of raw material and finished product:

| Working Capital (US \$ of 2015) |  |
| :---: | :---: |
| CURRENT ASSETS Inv. Raw Material (14 OD) |  |
|  |  |
| TM / DO | 78.7 |
| TM / Year | 1102.2 |
| MUS\$/TM | 1.17 |
| US\$/Year | 1,287,701 |
| Inv. Finished Products (14 DO) |  |
| TM / DO | 58.8 |
| TM / Year | 823.5 |
| MUS\$/TM | 2.13 |
| US\$/Year | 1,755,847 |
| Accounts Receivable (30 AD) |  |
| MUS\$/DC | 116.8 |
| MUS\$/Año | 3,504,822 |
| ACT. CURRENT (US \$ / Year) | 6,548,370 |
| CURRENT LIABILITIES <br> Accounts Payable (30 AD) |  |
|  |  |
| MUS\$/DC | 85.7 |
| US\$/Year | 2,570,363 |
| WORKING CAPITAL |  |
| US\$/Year | 3,978,007 |

## 7. BIBLIOGRAPHY

[1] Richard Turton: Analysis, Synthesis, and Design of Chemical Processes, 2nd ed., 958959.
[2] Treybal, R. E. "Mass Transfer Operations," 2nd ed., 1980. Cap. 12
[3] Perry, R. H., D. W. Green, J. O. Maloney (Editors) "Perry's Chemical Engineers' Handbook", 7th Ed. 1999. Ch. 12.
[4] Donald, Q. Kern. "Heat Transfer Processes", 3rd ed., 1980.
[5] Gavin Towler, Ray Sinnott. "Chemical Engineering Design".
[6] William L. Luyben. "Principles and Case Studies of Simultaneous Design".
[7] H. Scott Fogler. "Elements of Chemical Reaction Engineering".
8. ANNEXES

## ANNEX ${ }^{\circ}{ }^{1}$

## Mixer

| Residence time (min) | 30 |
| :--- | :---: |
| Mass to be stored (kg) | 1268.86 |
| Density (Kg / m3) | 780 |
| Useful volume (m3) | 1.6267 |
| Safety margin (\%) | 10 |
| Design Volume (m3) | 1.7894 |


| Dimensions of the mixer |  |
| :--- | :--- |
| Diameter of tank Dt (m) | 1.3500 |
| Impeller Diameter Da (m) | 0.4500 |
| Height of liquid H (m) | 1.3500 |
| Deflector thickness j (m) | 0.1350 |
| Tank free space E (m) | 0.4500 |
| Height of blade W (m) | 0.0900 |
| Shovel width L (m) | 0.1125 |
| Volume of tank V (m3) | 1.9324 |


| P heuristic (kW) | 0.19 |
| :---: | :---: |


| Determination of power for Re> 10000 |  |
| :--- | :---: |
| P (kW) | 0.30 |
| Fluid density (kg / m3) | 780.00 |
| Speed of rotation n (rps) | 1.25 |
| Impeller Diameter Da (m) | 0.5067 |
| KT | 5.80 |


| Viscosity (Kg / m.s) | 0.0015 |
| :--- | :---: |
| Reynolds number (Re) | 164234.47 |
| Velocidad de giro (m/s) | 0.6333 |

## ANNEX N ${ }^{\circ} 2$

## IPA Feed Pumps

From the following data obtained from the material balance.

| Current | $\mathbf{2}$ | $\mathbf{3}$ |  |  |
| :--- | :---: | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 33.470 | 33.470 |  |  |
| Pressure (kPa) | 99.80 | 400.00 |  |  |
| Fraction of vapor | 0 | 0 |  |  |
| Molar flow (kmol / hr) | 81.5159 | 81.5159 |  |  |
| Components |  |  |  |  |
| Molar flow (kmol / hr) |  |  |  |  |
| HYDROGEN | 2.0000 | 0.0000 |  |  |
| ACETONE | 26.67374857 | 26.67374857 |  |  |
| IPA | 3772.00 | 3772.00 |  |  |
| H20 | 777.6000 | 777.6000 |  |  |
| Mass flow (kg / h) | 0.00135 | 0.00135 |  |  |
| Density (kg / m3) |  |  |  |  |
| Flow rate (m3 / s) |  |  |  |  |

Performing energy balance at pump inlet and outlet:

$$
\mathrm{E}_{1}+\mathrm{H}_{\mathrm{B}}=\mathrm{E}_{2}
$$

$$
\begin{gathered}
\frac{\mathrm{P}_{1}}{\gamma}+\mathrm{Z}_{1}+\frac{\overline{\mathrm{V}}_{1}}{2 \mathrm{~g}}+\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}+\mathrm{Z}_{2}+\frac{\overline{\mathrm{V}}_{2}}{2 \mathrm{~g}} \\
\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}-\frac{\mathrm{P}_{1}}{\gamma}=\frac{300.2 \mathrm{kPa}}{9.81 \mathrm{~N} \times 777.6 \frac{\mathrm{~kg}}{\mathrm{~m}^{3}}}=39.354 \mathrm{~m}
\end{gathered}
$$

For the design, the data of the impeller diameter, roughness, and rotational speed are considered.

| Design data |  |
| :--- | :---: |
| Impeller Diameter Da (m) | 0.371 |
| Pump head H (m) | 39.3537 |
| Rotational speed n (1 / sec) | 1.4 |
| Power of pump P (W) | 404.5051971 |
| Discharge flow Q (m3 / s) | 0.00135 |
| Viscosity (Kg / m.s) | 0.001 |
| Density (Kg / m3) | 777.6000 |
| Roughness $\square$ (m) | 0.0001 |
| Coefficient of capacity Cq | 0.0188 |
| Coefficient of head Ch | 1431.036 |
| Power factor CP | 26.9721 |

## ANNEX N ${ }^{\circ} 3$

## Vaporizer

- Energy balance:

Of the following data:

## A. Heat balance.

For design considerations for a vaporizer, only $80 \%$ of the inlet flow to the schematic can be vaporized and since an outflow of $81.5159 \mathrm{kmol} / \mathrm{h}$ is required, the required flow rate of vaporizer inlet should be:

$$
\text { Flow rate of the vaporizer }=\frac{81.5159}{0.8}=101.8949 \mathrm{kmol} / \mathrm{h}
$$

The outline of the team is as follows:


As seen in the diagram, the inlet flow to the vaporizer is a mixture of the liquid stream coming down from the drum and line 3 , whereby, from an energy balance, the inlet temperature in the vaporizer is:

$$
\begin{gathered}
H_{\text {Mixture at } 33.47{ }^{\circ} \mathrm{C}} x \dot{m 3}+H_{\text {Mixture in } F . L . A t ~} 117.00^{\circ} \mathrm{C} \text { } x \dot{L} \\
=H_{\text {Mix in the feed to } T} x(m 3+L)
\end{gathered}
$$

From the energy balance, you get:

$$
T_{\text {feeding }}=76.08^{\circ} \mathrm{C}
$$

For the determination of the overall transfer coefficient, we have the following calculation sequence:

## B. Determination of water vapor flow.

Required heat in the Preheating zone: From subcooled liquid to saturated liquid.

- Enthalpy of the liquid mixture at $76.08^{\circ} \mathrm{C}: 57372.73 \mathrm{~kJ} / \mathrm{kmol}$
- Enthalpy of the liquid mixture at $117.00^{\circ} \mathrm{C}: 64095.21 \mathrm{~kJ} / \mathrm{kmol}$

$$
Q_{p}=(64095.21-57372.73) \times 101.8949=684986.43 \frac{\mathrm{~kJ}}{\mathrm{~h}}
$$

Heat required in the Vaporization zone: From saturated to saturated vapor.

- Steam enthalpy of the mixture at $117.00^{\circ} \mathrm{C}: 100865.01 \mathrm{~kJ} / \mathrm{kmol}$

$$
\begin{gathered}
Q_{v}=(100865.01-64095.21) \times 81.5159=2997323.34 \frac{\mathrm{~kJ}}{\mathrm{~h}} \\
Q=Q_{p}+Q_{v}=3682309.77 \mathrm{~kJ} / \mathrm{h}
\end{gathered}
$$

The required steam flow rate ( $165^{\circ} \mathrm{C}$ and 700 kPa ):

$$
\dot{m}_{\text {water vapor }}=\frac{3682309.77}{2072.959}=1776.35 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

## C. Correction of LMTD.

Since the input flow undergoes a phase change, the calculation of the LMTD will be performed in two zones:
a. Preheating zone:

$$
L M T D_{p}=\frac{(438.15 K-390.15 K)-(438.15 K-349.23 K)}{\ln \left(\frac{(438.15 K-390.15 K)}{(438.15 K-349.23 K)}\right)}=66.37 K
$$

b. Vaporization zone:

$$
L M T D_{v}=438.15 K-390.15 K=48.00 K
$$

c. Balanced LMTD:

$$
L M T D=\frac{Q_{p}+Q_{v}}{\frac{Q_{p}}{L M T D_{p}}+\frac{Q_{v}}{L M T D_{v}}}=\frac{3682309.77}{\frac{684986.43}{66.37}+\frac{2997323.34}{48.00}}=50.61 \mathrm{~K}
$$

## D. Determination of the global coefficient for the preheating zone.

- Coefficient of heat transfer per armor - ho.

For purposes of calculation the ho recommended by Donald Kern for steam flow will be used:

$$
h_{o}=8000 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

Internal area of a tube: $A_{t^{\prime}}=\frac{\pi x D i^{2}}{4}=0.00019 \mathrm{~m}^{2}$

Pipe area: $\quad A_{t}=\frac{N t \times A_{t \prime}}{\text { Number of steps }}=\frac{166 \times 0.00019}{6}=0.00536 \mathrm{~m}^{2}$

$$
\text { Mixing Flux: } \quad G_{t}=\frac{2537.72 \mathrm{~kg} / \mathrm{h}}{0.00536 \mathrm{~m}^{2}}=131.50 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}}
$$

Ha $T_{\text {average }}=\frac{76.08+117.70}{2}=96.54^{\circ} \mathrm{C}$, Properties of the fluid are:

$$
\mu=0.000361 \text { Pa.s } \quad C p=3.5333 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \quad k=0.2325 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{131.50 \times 0.0157 \mathrm{~m}}{0.000361 \mathrm{Pa.s}}=5728.98
$$

From Fig. 24 of Donald Kern's book "Heat Transfer Processes", Factor Jh is extracted for the given Reynolds number:

$$
J h=18.00
$$

So:

$$
h i=\operatorname{Jh} x \frac{k}{D e} x\left(\frac{C \mu}{k}\right)^{1 / 3}=469.7782 \frac{W}{m^{2} \cdot K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{8000 \times 469.7782}{8000+469.7782}=443.7218 \frac{W}{m^{2} \cdot K}
$$

Area required for preheating:

$$
A 1=\frac{\frac{684986.43}{66.37 \times 3.6}}{443.7218}=6.4609 \mathrm{~m}^{2}
$$

## E. Determination of the overall coefficient for the vaporization zone.

- Coefficient of heat transfer per armor - ho.

For purposes of calculation the ho recommended by Donald Kern for steam flow will be used:

$$
h_{o}=8000 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

$$
\text { Internal area of a tube: } \quad A_{t^{\prime}}=\frac{\pi x D i^{2}}{4}=0.00019 \mathrm{~m}^{2}
$$

Pipe area: $\quad A_{t}=\frac{N t \times A_{t}}{\text { Number of steps }}=\frac{166 \times 0.00019}{6}=0.00536 \mathrm{~m}^{2}$

$$
\text { Mixing Flux: } \quad G_{t}=\frac{2537.72 \mathrm{~kg} / \mathrm{h}}{0.00536 \mathrm{~m}^{2}}=131.50 \frac{\mathrm{~kg}}{\mathrm{s.m}^{2}}
$$

Ha $T_{\text {average }}=117.00^{\circ} \mathrm{C}$, l the properties of the fluid are:

$$
\begin{aligned}
\mu & =0.000255 \mathrm{~Pa} . \mathrm{s} \\
& =3.6698 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \\
C p k & =0.2253 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
\end{aligned}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{131.50 \times 0.0157 \mathrm{~m}}{0.000255 \text { Pa.s }}=8097.64
$$

From Fig. 24 of Donald Kern's book "Heat Transfer Processes", Factor Jh is extracted for the given Reynolds number:

$$
J h=29.00
$$

So:

$$
h i=\operatorname{Jh} \times \frac{k}{D e} \times\left(\frac{C \mu}{k}\right)^{1 / 3}=668.7653 \frac{W}{m^{2} \cdot K}
$$

For a clean global coefficient (U2):

$$
U 2=\frac{8000 \times 207.5479}{8000+207.5479}=617.1724 \frac{W}{m^{2} \cdot K}
$$

Area required for vaporization::

$$
A 2=\frac{\frac{2997323.34}{48 \times 3.6}}{617.1724}=28.1050 \mathrm{~m}^{2}
$$

## F. Checking maximum heat flow.

- Total clean area Ac:

$$
A c=A 1+A 2=6.4609+28.1050=34.5659 \mathrm{~m}^{2}
$$

- Total balanced clean coefficient:

$$
U=\frac{\frac{684986.43}{66.37 \times 3.6}+2997323.34 /(48 \times 3.6)}{34.5659}=584.75 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot \mathrm{~K}}
$$

- Total design coefficient:

$$
\begin{aligned}
\text { Total area }= & \text { Nt } \times L T \times R T=166 \times 6 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m} \\
& =59.5929 \mathrm{~m}^{2}
\end{aligned}
$$

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Ratio of pipe surface per linear foot (BWG 16)
From the following ratio ratio for the calculation of the design area for vaporization:

$$
\frac{28.1050}{34.5659} \times 59.5929 \mathrm{~m}^{2}=48.4541 \mathrm{~m}^{2}
$$

The heat flow in the vaporization zone is:

$$
Q_{F}=\frac{2997323.34}{48.4541 \times 3.6}=17183.06 \frac{\mathrm{~W}}{\mathrm{~m} 2}<78000 \frac{\mathrm{~W}}{\mathrm{~m} 2} \text { (Permissible heat flow) }
$$

If the heat flow in the vaporization zone is permissible, the conditions given for the next design are feasible.

## ANNEX ${ }^{\circ} \mathrm{C} 4$

## Preheater

Energy balance: From the following data:

## G. Heat balance.

The outline of the team is as follows:


For the determınation of the overall transfer coefficient, we have the following calculation sequence:

## H. Determination of the water vapor flow rate.

Required heat in the Preheating zone: From subcooled liquid to saturated liquid.

- Difference of enthalpies of the mixture in vapor phase from $117.00^{\circ} \mathrm{C}$ to $180.00^{\circ} \mathrm{C}$ : 179.8906 kJ / kg

$$
\begin{gathered}
Q_{p}=(179.8906) \times 2537.72=456511.97 \frac{\mathrm{~kJ}}{\mathrm{~h}} \\
Q=Q_{p}=456511.97 \mathrm{~kJ} / \mathrm{h}
\end{gathered}
$$

The required water vapor flow rate ( $1^{\circ} \mathrm{C}$ and 700 kPa ):

$$
\dot{m}_{\text {water vapor }}=\frac{456511.97}{1980.761}=230.4730 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

I. Determination of LMTD.

$$
L M T D=\frac{(471.44 K-453.15 K)-(471.44 K-390.15 K)}{\ln \left(\frac{(471.44 K-453.15 K)}{(471.44 K-390.15 K)}\right)}=42.24 K
$$

## J. Determination of the global coefficient for the heating zone.

- Coefficient of heat transfer per armor - ho.

For purposes of calculation the ho recommended by Donald Kern for steam flow will be used:

$$
h_{o}=8000 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

$$
\begin{aligned}
& \text { Internal area of a tube: } A_{t^{\prime}}=\frac{\pi x D i^{2}}{4}=0.00019 \mathrm{~m}^{2} \\
& \text { Pipe area: } A_{t}=\frac{\mathrm{Nt} x A_{t^{\prime}}}{\text { Number of steps }}=\frac{52 \times 0.00019}{1}=0.01008 \mathrm{~m}^{2} \\
& \text { Mixing Flux: } \quad G_{t}=\frac{2537.72 \mathrm{~kg} / \mathrm{h}}{0.01008 \mathrm{~m}^{2}}=69.96 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}}
\end{aligned}
$$

Ha $T_{\text {average }}=\frac{180.00+117.00}{2}=148.50^{\circ} \mathrm{C}$, Properties of the fluid are:

$$
\begin{aligned}
\mu & =0.000010 \mathrm{~Pa} . \mathrm{s} \\
C p & =1.9710 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \\
k & =0.0252 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
\end{aligned}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{69.96 \times 0.0157 \mathrm{~m}}{0.000010 \text { Pa.s }}=109812.20
$$

From Fig. 24 of Donald Kern's book "Heat Transfer Processes", Factor Jh is extracted for the given Reynolds number:

$$
J h=400.00
$$

So:

$$
h i=\operatorname{Jh} \times \frac{k}{D e} \times\left(\frac{C \mu}{k}\right)^{1 / 3}=369.4963 \frac{W}{m^{2} \cdot K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{8000 \times 369.4963}{8000+369.4963}=353.1838 \frac{W}{m^{2} \cdot K}
$$

Area required for heating:

$$
A 1=\frac{\frac{456511.97}{42.24 \times 3.6}}{353.1838}=8.5011 \mathrm{~m}^{2}
$$

- Total design coefficient:

$$
\begin{aligned}
\text { Total area }= & \text { Nt } \times L T \times R T=166 \times 6 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m} \\
& =9.3338 \mathrm{~m}^{2}
\end{aligned}
$$

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Ratio of pipe surface per linear foot (BWG 16)
So:

$$
U D=\frac{\frac{456511.97}{42.24 \times 3.6}}{9.3338}=321.64 \frac{W}{m^{2} \cdot K}
$$

## ANNEX Nํ 5

## Reactor

There is a tubular rectifier (PFR), in which the feed flow (F0) consists of IPA, water and acetone, and the outlet current (F), has IPA, water, DMK and H2. The characteristics of these currents are shown in the following table:

| Curretnt | $\mathbf{5}$ | $\mathbf{6}$ |
| :--- | :---: | :---: |
| Temperature $\left({ }^{\circ} \mathrm{C}\right)$ | 180 | 388 |
| Pressure (kPa) | 360 | 340 |
| Fraction of vapor | 1 | 1 |
| Molar flow (kmol / hr) | 81.52 | 128.95 |
| Components |  |  |
|  |  |  |
| Molar flow (kmol / hr) |  |  |
| H20 | 26.67 | 26.67 |
| IPA | 52.70 | 5.27 |
| HYDROGEN | 0.00 | 47.43 |
| ACETONE | 2.14 | 49.57 |

Applying the matter balance on a differential element of the axial position (dx) in the reactor for a tube, obtaining the following differential equation:

$$
\begin{gathered}
F_{A 0} \frac{d x_{A}}{\left(-r_{A}\right)}=d x * \frac{(1-e) * \pi * \mathrm{Di}^{\wedge} 2}{4} \\
\frac{d x_{A}}{d x}=\frac{\left(-r_{A}\right) *(1-e) * \pi * \mathrm{Di}^{\wedge} 2}{4 * F_{A 0}} \quad(A=I P A)
\end{gathered}
$$

For the reaction:

$$
\begin{aligned}
& I P A \rightarrow \text { Acetone }+\mathrm{H}_{2} \\
& H_{f}{ }^{\circ}=62900 \mathrm{~kJ} / \mathrm{kmol}
\end{aligned}
$$

$\mathrm{H}_{\mathrm{f}}{ }^{\circ}$ : Standard reaction heat.
The reaction rates are given by:

$$
\begin{gathered}
r_{F}=22 * 10^{6} \cdot C_{I P A} \cdot e^{-72380 / R T} \\
r_{R}=1000 \cdot C_{D M K} \cdot C_{H_{2}} \cdot e^{-9480 / R T} \\
r=\frac{\mathrm{kmol}}{\mathrm{~s} \cdot \mathrm{~m}^{3} \mathrm{cat}} \quad ; \quad C=\frac{\mathrm{kmol}}{\mathrm{~m}^{3} \mathrm{gas}} \quad ; \quad T=K
\end{gathered}
$$

The rate of reaction is determined by:

$$
\left(-r_{A}\right)=r=r_{D}-r_{I}=22 * 10^{6} \cdot C_{I P A} \cdot e^{-72380 / R T}-1000 \cdot C_{D M K} \cdot C_{H_{2}} \cdot e^{-9480 / R T}
$$

Applying the energy balance for a catalytic tubular reactor tube with heat exchange at steady state for a differential in the axial position (dx)
$\mathrm{d}(\mathrm{T}) / \mathrm{d}(\mathrm{x})=\left(\pi^{*} \mathrm{De}^{*} \mathrm{U}^{*}\left(\mathrm{~T}_{\text {oil }}-\mathrm{T}\right)+\Delta \mathrm{Hr}^{*} \mathrm{r}_{\mathrm{a}}{ }^{*}(1-\mathrm{e})^{*} \pi^{*} \mathrm{Di} \wedge 2 / 4\right) /\left(\mathrm{Fao} / \mathrm{N}^{*}\left(\Sigma \theta \mathrm{i}^{*} \mathrm{Cpi}+\mathrm{Xa} * \Delta \mathrm{Cp}.\right)\right)$
Where:
Fao: IPA molar flow entering the reactor.
N : number of tubes.
Di: Internal diameter.
From: External diameter.
U: Global heat transfer coefficient
Toil: Heater oil temperature.
T : Temperature inside the tubes.
E: Empty space inside the tubes.
$\Theta \mathrm{i}$ : Mole ratio of each component in the feed relative to the reactant (IPA).
$\Delta \mathrm{Hr}$ : reaction enthalpy as a function of reaction temperature.
Cpi: Heat capacity of component i.
Xa: Mole fraction of the feed

## X: Axial position

On the side of the heating oil, we have a temperature profile, which is found by applying a differential energy balance with respect to the axial position (dx), we obtain the following equation:

$$
\mathrm{d}\left(\mathrm{~T}_{\text {oil }}\right) / \mathrm{d}(\mathrm{x})=\mathrm{U}^{*} \pi^{*} \mathrm{De}^{*}\left(\mathrm{~T}-\mathrm{T}_{\text {oil }}\right) /\left(\mathrm{m}_{\text {oil }} * \mathrm{Cp}_{\text {oil }}\right)
$$

$\mathrm{m}_{\text {oil }}$ : Thermal oil mass flow.
Cpoil: Heat capacity of the thermal oil.
The thermal oil was selected Therminol VP-1, which supports a maximum temperature of $400^{\circ} \mathrm{C}$.

The characteristics of the oil as well as its properties are attached on the delivered CD.
The reaction enthalpy ( $\Delta \mathrm{Hr}$ ) is expressed by the following equation as a function of the reactor temperature along the tube:

$$
\Delta \mathrm{Hr}=\Delta \mathrm{Hr}^{\circ}+\Delta \mathrm{H}_{\mathrm{l}}+\lambda_{\text {vap. }}+\Delta \mathrm{H}_{\mathrm{v}}
$$

$$
\Delta \mathrm{H}_{\mathrm{l}}=\mathrm{f}\left(\mathrm{Cpl}_{i}\right) \quad \Delta \mathrm{Hv}=\mathrm{f}\left(\mathrm{Cpv}_{i}\right) \quad \Delta \mathrm{Hr}=\mathrm{f}\left(\mathrm{Tr}_{i}\right)
$$

The expressions for heat capacities are taken as a function of temperature for each component of the Perry's Chemical Engineers' Handbook [3].

For each component in liquid phase, the equation of the heat capacity is given by:

$$
\mathrm{Cpi}=\mathrm{C} 1+\mathrm{C} 2 * \mathrm{~T}^{*} \mathrm{C} 3^{*} \mathrm{~T}^{2}+\mathrm{C} 4 * \mathrm{~T}^{3}+\mathrm{C} 5 * \mathrm{~T}^{4}
$$

For each component (IPA, acetone) the constants of this equation are as follows:

| IPA |  | C3 | 36.662 |
| :---: | :---: | :---: | :---: |
| C1 | 723550 | C4 | -0.066395 |
| C2 | -8095 | C5 | 0.000044064 |


| Acetona |  | C3 | 0.2837 |
| :---: | :---: | :---: | :---: |
| C1 | 135600 | C4 | 0.000689 |
| C2 | -177 | C5 | 0 |

For each component (acetone, hydrogen, IPA and water) in vapor phase and / or gas:

CрAce $\left.=5.704^{*} 10^{\wedge 4+1.6320 * 10 \wedge 5 *((1.61 * 10 \wedge 3 / T) / S e n h(1.61 * 10 \wedge 3 / T)}\right)^{\wedge 2+9.680 * 10 \wedge 4 *((731.5 / T) / S e n h(731.5 / T)) \wedge 2 ~}$ $\left.\mathrm{CpH}_{2}=2.762^{*} 10 \wedge 4+9.56 * 10^{\wedge} 3^{*}\left(\left(2.466^{*} 10 \wedge 3 / T\right) / \operatorname{Senh}\left(2.466^{*} 10^{\wedge} 3 / T\right)\right)^{\wedge 2+3.76 * 10 \wedge 3 *((567.6 / T) / S e n h}(567.6 / T)\right)^{\wedge 2}$ CpIpa $=5.723^{*} 10 \wedge 4+1.91^{*} 10 \wedge 5^{*}\left(\left(1.4210^{*} 10 \wedge 3 / T\right) / \operatorname{Senh}\left(1.4210^{*} 10 \wedge 3 / T\right)\right)^{\wedge 2+1.2155 * 10 \wedge 5 *((626 / T) / S e n h ~(626 / T)) \wedge 2}$ $\mathrm{CpW}=3.336^{*} 10 \wedge 4+2.679^{*} 10^{\wedge} 4^{*}\left(\left(2.6105^{*} 10 \wedge 3 / \mathrm{T}\right) / \operatorname{Senh}\left(2.6105^{*} 10 \wedge 3 / \mathrm{T}\right)\right)^{\wedge} 2+8.9^{*} 10^{\wedge} 3^{*}((1169 / \mathrm{T}) / \operatorname{Senh}(1169 / \mathrm{T})) \wedge 2$

The enthalpy of evaporation for each component (IPA and acetone) is given by the following equation:

$$
\lambda \text { vap. }=\mathrm{C} 1 *(\mathrm{Tr})^{\wedge}(\mathrm{C} 2+\mathrm{C} 3 * \operatorname{Tr}+\mathrm{C} 4 * \operatorname{Tr} \wedge 2)
$$

IPA

| C1 | 63080000 | C3 | 0 |
| :--- | :--- | :--- | :--- |
| C2 | 0.3921 | C4 | 0 |

## ACETONE

| C1 | 42150000 | C3 | 0 |
| :--- | :--- | :--- | :--- |
| C2 | 0.3397 | C4 | 0 |

Where:
Tri: reduced temperature of component i, (Tebulli / Tcriticoi)
Tebulli: Boiling temperature of component i .
Tcriticoi: Critical temperature of component i.
For each component, the following data is available:

| Component | Molar flow <br> (kmol / hr) | Xi | $\theta$ |
| :---: | :---: | :---: | :---: |
| HYDROGEN | 0 | 0.0000 | 0 |
| ACETONE | 0.03 | 0.0263 | 0.0407 |
| IPA | 53.17 | 0.6465 | 1 |
| H20 | 24.368 | 0.3272 | 0.506 |

In order to solve the differential equations we used the Polymath program 6.10 (the calculations and results obtained are appended in the CD), which is described in the literature [7], for a heat transfer coefficient equal to $80 \mathrm{Wm}-2 \mathrm{~K}-1$, the input temperature was taken to be $180^{\circ} \mathrm{C}$, which is the temperature at which it leaves the preheater and the reactor output is calculated to have a temperature of 660.65 K with a conversion per step of 0.90 .

The temperature profiles inside the reactor and the heating oil are obtained with the program and are plotted obtaining:


The conversion as a function of the axial position has the following graph:



The catalyst occupies a certain volume inside the reactor tubes which can be found by means of the attached graph, obtaining an empty space equal to 0.36 for compact cylinders.

The use of a fixed bed reactor with the catalyst is proposed in 0.0254 m ( 1 in ) outer diameter tubes. It is estimated that the length of the tubes will be $6,096 \mathrm{~m}$, resulting in a number of tubes equal to 490.

With the total of tubes, the volume of these in addition to the empty space can be found the mass of catalyst to be used.

$$
M_{c a t}=N * V_{\text {tubo }} *(1-e) * \rho_{\text {real }}=0.519 \mathrm{~m}^{3} * 2000 \frac{\mathrm{~kg}}{\mathrm{~m}^{3}}=1038.08 \mathrm{~kg}
$$

The total heat exchange area is determined by the number of tubes and the external area of the tube.

$$
A_{\text {total }}=N * A_{\text {tube }}=490 * \pi * D e * L=238.36 \mathrm{~m} 2
$$

The power needed to reach this conversion is given by:

$$
Q_{\text {req. }}=U * \overline{C p} *\left(T a_{o}-T a_{f}\right)=0.08 * 2.62798 *(400-396.6)=1.244 \mathrm{MW}
$$

## ANNEX Nํ 6

## Thermal oil furnace

The heat requirement for the thermal oil found in the reactor design is of the order of 1,244 MW but in the furnace there are heat losses in the walls of the furnace, an acceptable value is $2 \%$ and also there are losses by gases produced by The combustion (H2O, CO2, N 2 and O 2 in excess) which gain heat reducing the heat yielded by the combustion, there are correlations between the efficiency of the furnace and the temperature of the output in the chimney, thus we have:

$$
\begin{aligned}
& T_{\text {stack }}=\left\{\left(0.98-E_{\text {ff }}\right) / 9.25 \times 10^{-5}\left(1+\frac{e x_{\text {air }}}{100}\right)^{0.748}\right\}^{1 / 1.128} \\
& E \text { ff }=0.98-9.25 \times 10^{-5} T_{\text {stack }} 1128\left(1+\frac{e x_{\text {air }}}{100}\right)^{0.748}
\end{aligned}
$$

Where:

Tstack: temperature at the entrance of the chimney.

Eff: Oven efficiency

The outlet temperature of the gases is in the order of 25 to $40^{\circ} \mathrm{C}$ higher than the inlet fluid temperature.

The incoming furnace enters a temperature of $388^{\circ} \mathrm{C}$; therefore, the temperature of the exit of the gases would be, taking a value of $420^{\circ} \mathrm{C}\left(788^{\circ} \mathrm{F}\right)$, when in addition the efficiency would be given by:

Eff: 0.896

The required heat in the oven to meet the requirement would be:

$$
Q_{\text {horno }}=1.38815 \mathrm{MW}
$$

## ANNEX N ${ }^{\circ} 7$

## Cooler

From the following data:

| Current | Inlet | outlet | L1 | L2 |
| :--- | :---: | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 387.65 | 92.17 | 21.85 | 40.00 |
| Temperature (K) | 660.80 | 365.32 | 295.00 | 313.15 |
| Pressure (kPa) | 320.00 | 300.00 | 300.00 | 300.00 |
| Fraction of vapor | 1.00 | 1.00 | 0.00 | 0.00 |
| Molar flow (kmol / hr) | 128.9453 | 128.9453 | 2066.9000 | 2066.9000 |
| Components |  |  |  |  |
| Molar flow (kmol / hr) |  |  |  |  |
| H20 | 26.6737 | 26.6737 | 2066.9000 | 2066.9000 |
| IPA | 5.2699 | 5.2699 | 0.0000 | 0.0000 |
| HYDROGEN | 47.4294 | 47.4294 | 0.0000 | 0.0000 |
| ACETONE | 49.5722 | 49.5722 | 0.0000 | 0.0000 |
| Molar fractions |  |  |  |  |
| H20 | 0.2069 | 0.2069 | 1.000 | 1.000 |
| IPA | 0.0409 | 0.0409 | 0.000 | 0.000 |
| HYDROGEN | 0.3678 | 0.3678 | 0.000 | 0.000 |
| ACETONE | 0.3844 | 0.3844 | 0.000 | 0.000 |
| Mass flow (kg / h) | 3772.00 | 3772.0042 | 37235.4823 | 37235.4823 |

## K. Determination of the water vapor flow rate.

The required heat for inlet flow condensation is:

$$
Q_{p}=2490000 \frac{\mathrm{~kJ}}{\mathrm{~h}}
$$

The required cooling water flow rate ( $165^{\circ} \mathrm{C}$ y 700 kPa ):

$$
\dot{m}_{\text {water vapor }}=\frac{2490000}{66.8717}=37235.48227 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

L. Determination of LMTD.

$$
L M T D=\frac{(298.15 K-295.00 K)-(313.15 K-660.80 K)}{\ln \left(\frac{(298.15 K-205.00 K)}{(313.15 K-660.80 K)}\right)}=173.53 K
$$

## M. Determination of the overall coefficient.

- Coefficient of heat transfer per armor - ho.

Outer tube diameter: $\quad D_{o}=0.0191 \mathrm{~m}^{2}$
Separation between Pitch and Do : $\quad C^{\prime}=0.0064 \mathrm{~m}^{2}$
Armor Diameter: $\quad D_{c}=0.3048 \mathrm{~m}(12 \mathrm{pulg}$.
Space between baffles: $B=0.0610 \mathrm{~m}$
Armored area: $\quad$ As $=\frac{0.0064 \times 0.3048 \times 0.0610}{0.0254}=0.0047 \mathrm{~m}^{2}$
Water mass flow: $\quad G_{t}=\frac{37235.48 \frac{\mathrm{~kg}}{\mathrm{~h} . \mathrm{m2}}}{0.0047 \mathrm{~m}^{2} \times 3600}=2209.268 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}}$
Taverage $=30.93^{\circ} \mathrm{C}$, the properties of the cooling water are:

$$
\begin{aligned}
\mu & =0.00076 \mathrm{~Pa} \cdot \mathrm{~s} \\
C p & =4.2267 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \\
k & =0.6217 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
\end{aligned}
$$

The equivalent diameter of the armor:

$$
\operatorname{De}=4 x\left(\frac{0.8661 x\left(\left(\left(\frac{0.0254}{0.0254}\right)\right)^{2}-\frac{\pi}{4} x\left(\frac{0.0191}{0.0254}\right)^{2}\right)}{\pi x\left(\frac{0.0191}{0.0254}\right)}\right) x 0.0254=0.01815 \mathrm{~m}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{2209.268 \times 0.01815 \mathrm{~m}}{0.00076 \text { Pa. } \mathrm{s}}=53441.54
$$

From Fig. 28 of Donald Kern's book "Heat Transfer Processes", the Factor Jh is extracted for the given Reynolds number:

$$
J h=135.00
$$

So:

$$
H o=\operatorname{Jh} x \frac{k}{D e} \times\left(\frac{C \mu}{k}\right)^{1 / 3}=22901.2898 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

$$
\text { Internal area of a tube: } \quad A_{t}=\frac{\pi x D i^{2}}{4}=0.00019 \mathrm{~m}^{2}
$$

$$
\begin{gathered}
\text { Pipe area: } A_{t}=\frac{N t \times A_{t}}{\text { Number of steps }}=\frac{82 \times 0.00019}{1}=0.01589 \mathrm{~m}^{2} \\
\text { Mixing Flux: } \quad G_{t}=\frac{3772 \mathrm{~kg} / \mathrm{h}}{0.01589 \mathrm{~m}^{2}}=65.95 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}} \\
T_{\text {average }}=\frac{387.65+92.17}{2}=239.91^{\circ} \mathrm{C}, \text { Properties of the fluid are: } \\
\mu=0.000017 \mathrm{Pa.s} \\
C p=4.0566 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \\
k=0.0370 \frac{\mathrm{~W}}{\mathrm{~m} . \mathrm{K}}
\end{gathered}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{65.95 \times 0.0157 \mathrm{~m}}{0.000017 \text { Pa.s }}=60705.38
$$

From Fig. 24 of Donald Kern's book "Heat Transfer Processes", Factor Jh is extracted for the given Reynolds number:


$$
J h=160.00
$$

So:

$$
h i=\operatorname{Jh} x \frac{k}{D e} x\left(\frac{C \mu}{k}\right)^{1 / 3}=464.7955 \frac{W}{m^{2} \cdot K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{22901.2898 \times 464.7955}{22901.2898+464.7955}=455.5498 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot K}
$$

Area required for heating:

$$
A 1=\frac{\frac{2490000}{170.61 \times 3.6}}{455.5498}=8.8993 \mathrm{~m}^{2}
$$

## N. Checking maximum flow of heat.

- Total clean area Ac:

$$
A c=A 1=8.8993 \mathrm{~m}^{2}
$$

- Total design coefficient:

Total area $=N t \times L T \times R T=82 \times 3 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m}=$ $14.7187 \mathrm{~m}^{2}$ :

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Tube surface ratio per linear foot (BWG 16)

$$
U=\frac{\frac{2490000}{170.61 \times 3.6}}{14.7187}=275.4374 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot K}
$$

## ANNEX ${ }^{\circ} 8$

## Phase separator

In a phase equilibrium with multicomponent system, the equilibrium in the vapor phase and the liquid phase following the relationship:

$$
y_{i}=K_{i} * x_{i}
$$

Where Ki is a function of the drum pressure as well as the temperature and compositions.
Since it is a non-ideal mixture it is necessary to use the Ki corrected with the activity and fugacity of each component, for which a thermodynamic model is used, the model to be used for this kind of non-ideal mixture is the UNIQUAC package.

The conditions at which the separator operates are 300 kPa and $32{ }^{\circ} \mathrm{C}$, the equilibrium constants (Ki) are using the Aspen Hysys 8.8 simulator database, using the aforementioned thermodynamic package; for the feed stream defined by pressure, temperature and composition, the following Ki values were obtained.

| EQUILIBRIUM DATA (Ptamb., Ttamb.) |  |
| :--- | :---: |
| K |  |
| Hydrogen | 2191.73 |
| Acetone | 0.17 |
| IPA | 0.05 |
| Water | 0.04 |

Now for the phase equilibrium in the separation column, by means of a material balance we obtain that:

$$
F * z_{i}=L * x_{i}+V * y_{i}
$$

Replacing the relationship between the molar fractions of the vapor and liquid the balance equation will take the following form:

$$
F * z_{i}=L * x_{i}+V * K_{i} * x_{i}
$$

Xi is cleared and knowing that $\mathrm{L}=\mathrm{F}-\mathrm{V}$, the expression would remain:

$$
x_{i}=\frac{z_{i}}{1+\left(K_{i}-1\right) * \frac{V}{F}}
$$

Where: V / F is the vaporized fraction.
For both phases you have to satisfy that the sum of your mole fractions are equal to 1 is why if we subtract both expressions we would have:

$$
f\left(\frac{V}{F}\right)=\sum_{i=1}^{c} \frac{\left(K_{i}-1\right) * z_{i}}{1+\left(K_{i}-1\right) * \frac{V}{F}}=0
$$

This expression is called; Rachford-Rice's equation to find this the vaporized fraction must be proceeded by a numerical method for its calculation.

For the following feeding conditions:

| MOLAR FEED FLOW |  |
| :---: | :---: |
| $\mathrm{F}(\mathrm{kmol} / \mathrm{h})$ | 128.95 |
| Composition | $\mathbf{Z}_{\mathbf{f}}$ |
| Hydrogen | 0.368 |
| Acetone | 0.384 |
| IPA | 0.041 |
| Water | 0.207 |

Solving the above equation is the vaporized fraction and the compositions in the liquid and vapor, the vaporized fraction is obtained by the convergence of the equation presented above.

## Balance de materia:

|  | V/F | 0.415 |
| :---: | :---: | :---: |
|  | MOLAR LIQUID <br> FLOW | MOLAR OF STEAM <br> FLOW |
|  | 75.45 | 53.50 |
|  | MOLAR LIQUID <br> COMPOSITION | MOLAR VAPOR <br> COMPOSITION |
| Hydrogen | 0.0004 | 0.8860 |
| Acetone | 0.5871 | 0.0986 |
| IPA | 0.0675 | 0.0033 |
| Water | 0.3449 | 0.0121 |

With the calculated values, proceed to the design of the vertical drum of the separator using the following empirical equations:

Maximum permissible speed - Uperm. (Feet): $\quad u_{\text {perm }}=\mathrm{K}_{\text {tamb }} \sqrt{\frac{\rho_{\mathrm{L}}-\rho_{\mathrm{v}}}{\rho_{\mathrm{v}}}}$

This velocity is found by the relation of the densities of the phases as well as the drum type, represented by Ktamb, by means of the Blackwell correlation, according to the following values:
$\mathrm{K}_{\text {tamb }}=\exp \left[\mathrm{A}+\mathrm{B} \ln \mathrm{F}_{\mathrm{lv}}+\mathrm{C}\left(\ln \mathrm{F}_{\mathrm{w}}\right)^{2}+\mathrm{D}\left(\ln \mathrm{F}_{\mathrm{lv}}\right)^{3}+\mathrm{E}\left(\ln \mathrm{F}_{\mathrm{lv}}\right)^{4}\right] \quad$ donde $\mathrm{F}_{\mathrm{lv}}=\frac{\mathrm{W}_{\mathrm{L}}}{W_{v}} \sqrt{\frac{\rho_{v}}{\rho_{\mathrm{L}}}}$

| WHERE |  |
| :---: | :---: |
| A | -1.8774781 |
| B | -0.81458046 |
| C | -0.18707441 |
| D | -0.01452287 |
| E | -0.00101485 |

Calculate the values to find the Ktamb.

| Wliq | 1521.8 |
| :---: | :---: |
| Wvap | 192.8 |
| Fjv | 0.270 |

Where:
Wliq: Liquid mass flow
Wvap: Steam mass flow

|  | feet/s | $\mathrm{m} / \mathrm{s}$ |
| :---: | :---: | :---: |
| Uperm | 9.705 | 2.958 |
| Ktamb | 0.332 | 0.101 |

The maximum permissible speed is transformed to find the transverse area:

$$
\mathrm{V}=\frac{\mathrm{U}_{\mathrm{perm}} * 3600 * \mathrm{~A}_{\mathrm{c}} * \rho_{\mathrm{v}}}{\mathrm{PM}_{\mathrm{V}}}
$$

$$
A_{c}=\frac{\mathrm{PM}_{\mathrm{V}} * V}{\mathrm{U}_{\mathrm{perm}} * 3600 * \rho_{\mathrm{v}}}
$$

| Transverse area (Ac) | $0.0425 \mathrm{~m}^{2}$ |
| :--- | :--- |

The diameter for the vertical drum is obtained:

$$
\begin{array}{|l|l|}
\hline \text { Diameter of drum (D) } & 0.233 \mathrm{~m} \\
\hline
\end{array}
$$

Choosing a commercial diameter value of 10 "nominal diameter, 80, its internal diameter is 0.243 m .

Establish the length / diameter ratio, either by an approximate rule or with the volume necessary to contain strokes of liquid flow. For vertical flash distillation drums, the approximate rule is that hf / D is from 3.0 to 5.0. The appropriate value of $\mathrm{hf} / \mathrm{D}$ within this range can be relative to the pressure in the separation tank, for design pressure the L / D should take the value of 5 .

The drum height above the feed nozzle shaft, hv, should be 36 in . Plus half the diameter of the feed tube.

The depth of liquid hL can be determined from the desired reserve volume, V .

$$
\mathrm{h}_{\mathrm{L}}=\frac{4 * F v a p * t r}{\pi * \mathrm{Di}^{2}}
$$

Dónde:
Fvap: Volumetric flow of the liquid ( $\mathrm{m}^{3} / \mathrm{h}$ )

## Tr : Residence time

With an hf / D ratio of 5 and a residence time of 2 minutes, the following results are obtained:

| DRUM HEIGHT |  |
| :--- | :---: |
| D feed (m) | 0.02540 |
| HV (m) | 0.826 |
| Hf (m) | 1.214 |
| HL (m) | 3.008 |
| Htotal (m) | 5.048 |

With the dimensions of the phase separating tank of 0.243 m diameter by 5.048 m in height, and for the nominal diameter 10 " 80 schedule, the thickness of the tower is 1.51 cm , in addition to the density of stainless steel 304 which is $7.9 \mathrm{~g} / \mathrm{cm} 3$, the weight of the separator with which the equipment is supplied is obtained by equation:

$$
\begin{gathered}
\mathrm{m}_{\text {armor }}=\pi * \mathrm{Di} * \mathrm{H}_{\text {total }} * \mathrm{e} * \rho_{\text {steel }} \\
\mathrm{m}_{\text {armor }}=459.08 \mathrm{Kg}
\end{gathered}
$$

## ANNEX N ${ }^{\circ} 9$

## Distillation Tower of Acetone

- Knowing the composition of the feed stream (F), we define acetone as light key (LK) and isopropyl alcohol (IPA) as heavy key (HK), whose compositions depend on the separation to be made.

| Molar composition and molar fluxes of DISTILLATE and FUNDS |  |  |  |  |  |
| :--- | :---: | :---: | :---: | :---: | :---: |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{D}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{B}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{x D}$ | $\mathbf{x B}$ |
| Hydrogen | 0.0305 | 0.0305 | 0.0000 | 0.0007 | 0.0000 |
| Acetone | 44.2993 | 42.1564 | 2.1429 | 0.9983 | 0.0645 |
| IPA | 5.0940 | 0.0422 | 5.0518 | 0.0010 | 0.1521 |
| Water | 26.0252 | 0.0000 | 26.0252 | 0.0000 | 0.7834 |
| TOTAL | $\mathbf{7 5 . 4 4 9 0}$ | $\mathbf{4 2 . 2 2 9 1}$ | $\mathbf{3 3 . 2 1 9 9}$ | $\mathbf{1 . 0 0 0 0}$ | $\mathbf{1 . 0 0 0 0}$ |

- We determine the dew and bubble temperature of the 3 currents entering and leaving the tower, as they will be necessary in the design of the condenser and reboiler. The procedure to be performed for both the spray point and bubble is the same for any current, in this calculation will be used the values of distribution coefficients (Ki) of the Aspen Hysys simulator database for each current defined by its pressure, Temperature and composition.
- Proceeding in the same way with the other currents you get:

| CURRENT | $\mathbf{P ( k P a )}$ | $\left.\mathbf{T b u r b}^{\left({ }^{\circ} \mathbf{C}\right.}\right)$ | Trocio $^{\left({ }^{\circ} \mathbf{C}\right)}$ |
| :--- | :---: | :---: | :---: |
| Feeding | 300 | 38.20 | 103.00 |
| Distilled | 280 | 45.78 | 89.25 |
| Money | 300 | 107.00 | 125.90 |

- We applied the ShortCut Method to determine the number of ideal dishes (Nid) for a reflux ( $\mathrm{R}=1.25 \mathrm{Rm}$ ), the feed plate ( NF ), the minimum number of plates $(\mathrm{Nm})$ and minimum reflux tower. The first step is to verify if the components are distributed as initially assumed, for this the Shiras equation:

| Bubble Point supply |  |  | Temperature $\left({ }^{\circ} \mathbf{C}\right)$ |  | Pressure (kPa) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| It is iterated until: | $\mathbf{1} / \sum \boldsymbol{\alpha} \cdot \mathbf{x i}-\mathbf{K r}$ | 0.0000 | 38.20 |  | 300 |  |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{x i}$ | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\boldsymbol{\alpha} . \mathbf{x i}$ | $\mathbf{y i}$ |
| Hydrogen | 0.0305 | 0.0004 | 2111 | 30869.84 | 12.4794 | 0.8532 |
| Acetone | 44.2993 | 0.5871 | 0.2134 | 3.12 | 1.8326 | 0.1253 |
| IPA | 5.0940 | 0.0675 | 0.0684 | 1.00 | 0.0675 | 0.0046 |
| Water | 26.0252 | 0.3449 | 0.0490 | 0.72 | 0.2471 | 0.0169 |
| TOTAL | 75.4490 | $\mathbf{1 . 0 0 0 0}$ |  |  | $\mathbf{1 4 . 6 2 6 6}$ | $\mathbf{1 . 0 0 0 0}$ |


| Dew Point supply |  |  | Temperature $\left({ }^{\circ} \mathbf{C}\right)$ |  | Pressure (kPa) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| It is iterated until: | $\sum(\mathbf{y i} / \boldsymbol{\alpha})-\mathbf{K r}$ | 0.0000 | 103.00 |  | 300 |  |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{y i}$ | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\mathbf{y i} / \boldsymbol{\alpha}$ | $\mathbf{x i}$ |
| Hydrogen | 0.0305 | 0.0004 | 11797 | 3718.58 | 0.0000 | 0.0000 |
| Acetone | 44.2993 | 0.5871 | 5.3974 | 1.70 | 0.3451 | 0.1088 |
| IPA | 5.0940 | 0.0675 | 3.1725 | 1.00 | 0.0675 | 0.0213 |
| Water | 26.0252 | 0.3449 | 0.3965 | 0.12 | 2.7599 | 0.8699 |
| TOTAL | 75.4490 | $\mathbf{1 . 0 0 0 0}$ |  |  | $\mathbf{3 . 1 7 2 5}$ | $\mathbf{1 . 0 0 0 0}$ |

$$
\frac{\mathrm{x}_{\mathrm{JD}} \mathrm{D}}{\mathrm{z}_{\mathrm{JF}} \mathrm{~F}}=\left(\frac{\alpha_{\mathrm{J}}-1}{\alpha_{\mathrm{lk}}-1}\right) \frac{\mathrm{x}_{\mathrm{lkD}} \mathrm{D}}{\mathrm{z}_{\mathrm{lkF}} \mathrm{~F}}+\left(\frac{\alpha_{\mathrm{lk}}-\alpha_{\mathrm{J}}}{\alpha_{\mathrm{lk}}-1}\right) \frac{\mathrm{x}_{\mathrm{hkD}} \mathrm{D}}{\mathrm{z}_{\mathrm{hkF}} \mathrm{~F}}
$$

- A component will be distributed if $0 \leq \mathrm{xJDD} / \mathrm{zJFF} \leq 1$, therefore:
- We calculate the minimum backflow (Rm) using the Underwood method, for this it is necessary to determine the thermal condition of the feed mixture ("q") and the value of $\varphi$ obtained by iterating for values between the $\alpha$ Components.

| Enthalpy of the feeding |  |
| :---: | :---: |
| Temperature $\left({ }^{\circ} \mathbf{C}\right)$ | 32 |
| $\mathbf{H}_{\mathbf{F}}(\mathbf{k J} / \mathbf{k m o l})$ | $-2.641 \mathrm{E}+05$ |
| $\mathbf{H}_{\mathbf{G}}(\mathbf{k J} / \mathbf{k m o l})$ | $-2.242 \mathrm{E}+05$ |
| $\mathbf{H}_{\mathbf{L}}(\mathbf{k J} / \mathbf{k m o l})$ | $-2.648 \mathrm{E}+05$ |

$$
\begin{gathered}
\mathrm{q}=\frac{\mathrm{H}_{\mathrm{G}}-\mathrm{H}_{\mathrm{F}}}{\mathrm{H}_{\mathrm{G}}-\mathrm{H}_{\mathrm{L}}} \\
\mathrm{q}=\frac{-2.242-(-2.641)}{-2.242-(-2.648)}=1.0173
\end{gathered}
$$

| Key component check |  |  |  |  | Tav ( ${ }^{\circ} \mathrm{C}$ ) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| We assume $\mathbf{K i}=\left[(\mathrm{Ki})_{\text {top }} \text {. }(\mathrm{Ki})_{\text {bot }}\right]^{0.5}$ |  |  |  |  |  |  |
| Components | F (kmol/h) | D (kmol/h) | Ki | $\alpha=\mathbf{K i} / \mathbf{K r}$ | X ${ }_{\text {Jd. }}$ D/Z $\mathrm{Z}_{\text {JF. }}$.F | Distribute? |
| Hydrogen | 0.0305 | 0.0305 | 2710 | 1964.04 | 3657.60 | No |
| Acetone | 44.2993 | 42.1564 | 2.08 | 1.51 | 0.9516 | Si |
| IPA | 5.0940 | 0.0422 | 1.38 | 1.00 | 0.0083 | Si |
| Water | 26.0252 | 0.0000 | 0.72 | 0.52 | -0.8786 | No |
| TOTAL | 75.4490 | 42.2291 |  |  |  |  |

- The values of q and $\varphi$ are replaced in the following equations:

$$
\sum \frac{\alpha_{\mathrm{J}} \mathrm{Z}_{\mathrm{JF}} \mathrm{~F}}{\alpha_{\mathrm{J}}-\varphi}=\mathrm{F}(1-q) \quad \sum \frac{\alpha_{\mathrm{J}} \mathrm{X}_{\mathrm{JD}} \mathrm{D}}{\alpha_{\mathrm{J}}-\varphi}=\mathrm{D}\left(\mathrm{R}_{\mathrm{m}}+1\right)
$$

| Calculation of the minimum reflux (Rm) |  |  |  | $\boldsymbol{\varphi}=$ |
| :---: | :---: | :---: | :---: | :---: |
| For supply q $=$ | 1.0173 | $\mathbf{1}-\mathbf{q}-\sum \boldsymbol{\alpha} \cdot \mathbf{z F} /(\boldsymbol{\alpha}-\boldsymbol{\varphi})=$ |  | 0.0428 |
| Components | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\boldsymbol{\alpha} \cdot \mathbf{z F} /(\boldsymbol{\alpha}-\boldsymbol{\varphi})$ | $[\boldsymbol{\alpha} . \mathbf{x D} /(\boldsymbol{\alpha}-\boldsymbol{\varphi})] \cdot \mathbf{D}$ |
| Hydrogen | 2710 | 1964.04 | 0.0004 | 0.0305 |
| Acetone | 2.08 | 1.51 | 1.9081 | 137.0033 |
| IPA | 1.38 | 1.00 | -1.5774 | -0.9859 |
| Water | 0.72 | 0.52 | -0.3484 | 0.0000 |
| TOTAL |  |  | $\mathbf{- 0 . 0 1 7 3}$ | $\mathbf{1 3 6 . 0 4 7 9}$ |


| $\mathbf{D}(\mathbf{R m}+\mathbf{1})$ | 136.0479 |
| :---: | :---: |
| $\mathbf{R m}$ | $\mathbf{2 . 2 2 1 7}$ |

- The minimum number of plates (Nm) is determined by the equation of Fenske:

$$
\mathrm{N}_{\mathrm{m}}=\frac{\log \left[\left(\frac{\mathrm{x}_{\mathrm{lk}}}{\mathrm{x}_{\mathrm{hk}}}\right)_{\mathrm{D}}\left(\frac{\mathrm{x}_{\mathrm{hk}}}{\mathrm{x}_{\mathrm{lk}}}\right)_{\mathrm{B}}\right]}{\log \left(\alpha_{\mathrm{lkAV}}\right)}-1 \quad \begin{array}{|c|c|}
\hline \boldsymbol{\alpha l k A V} & 1.51 \\
\hline \mathbf{N m} & \mathbf{1 7 . 9 5} \\
\hline
\end{array}
$$

$$
\alpha_{\mathrm{lkAV}}=\sqrt{\left(\frac{\mathrm{K}_{\mathrm{lk}}}{\mathrm{~K}_{\mathrm{hk}}}\right)_{\mathrm{D}}\left(\frac{\mathrm{~K}_{\mathrm{lk}}}{\mathrm{~K}_{\mathrm{hk}}}\right)_{\mathrm{B}}}
$$

- The number of ideal dishes (Nid) for a given reflux ratio ( $\mathrm{R}=1.25 \mathrm{Rm}$ ) is determined by the correlation of Gilliland:
$X=\frac{R-R_{m}}{R+1}$
$\frac{\mathrm{N}-\mathrm{N}_{\mathrm{m}}}{\mathrm{N}+1}=1-\mathrm{e}^{\left[\frac{1+54.4 \mathrm{X}}{11+117.2 \mathrm{X}}\left(\frac{\mathrm{X}-1}{\mathrm{X}^{0.5}}\right)\right]}$

| $\mathbf{R} / \mathbf{R m}$ | 1.25 |
| :---: | :---: |
| $\mathbf{R}$ | 2.78 |
| $\mathbf{X}=\mathbf{( R}-\mathbf{R m}) /(\mathbf{R}+\mathbf{1})$ | 0.15 |
| $\mathbf{( N - N m )} /(\mathbf{N}+\mathbf{1})$ | 0.51 |
| $\mathbf{N i d}$ | $\mathbf{3 7 . 5 1}$ |

- The Kirkbride equation gives an approximate location of the feed plate:

$$
\frac{N_{\mathrm{R}}}{\mathrm{~N}_{\mathrm{S}}}=\left[\left(\frac{\mathrm{Z}_{\mathrm{hkF}}}{\mathrm{z}_{\mathrm{lkF}}}\right)\left(\frac{\mathrm{x}_{\mathrm{lkB}}}{\mathrm{x}_{\mathrm{hkD}}}\right)\left(\frac{\mathrm{B}}{\mathrm{D}}\right)\right]^{0.206}
$$

| NR/NS | 3.39 |
| :---: | :---: |
| NR | $\mathbf{2 8 . 9 7}$ |
| NS | $\mathbf{8 . 5 4}$ |

- For the design of perforated plates of cross-flow of a single step was taken into consideration the maximum flows of both liquid and vapor in the tower located in the area of impoverishment (bottom of the tower), in addition were used the equations of the book "Mass Transfer Operations - Robert E. Treybal, Chapter 6".

| COMPOSITION AND MOLAR FLUX OF LIQUID |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| $\mathbf{L ( k m o l} / \mathbf{s})$ |  | 0.0535 |  |  |
| $\mathbf{N}^{\mathbf{o}}$ | COMPONENTS | $\mathbf{M ~ ( k g / k m o l})$ | $\mathbf{x i}$ | $\mathbf{L i}$ <br> $(\mathbf{k m o l} / \mathbf{h})$ |
| 1 | Hydrogen | 2.016 | 0.0000 | 0.0000 |
| 2 | Acetone | 58.080 | 0.8378 | 161.3709 |
| 3 | IPA | 60.090 | 0.0271 | 5.2112 |
| 4 | Water | 18.016 | 0.1351 | 26.0252 |
| Maverage (kg/kmol) |  | $\mathbf{5 2 . 7 2}$ | $\mathbf{5 2 . 7 2}$ | $\mathbf{1 9 2 . 6 0 7 3}$ |

## COMPOSITION AND MOLAR FLOW OF STEAM

G (kmol/s)

| $\mathbf{N}^{\mathbf{o}}$ | COMPONENTS | $\mathbf{M ( k g / k m o l})$ | $\mathbf{y i}$ | $\mathbf{G i}$ <br> $(\mathbf{k m o l} / \mathbf{h})$ |
| :---: | :---: | :---: | :---: | :---: |
| 1 | Hydrogen | 2.016 | 0.0000 | 0.0000 |
| 2 | Acetone | 58.080 | 0.9990 | 159.2280 |
| 3 | IPA | 60.090 | 0.0010 | 0.1594 |
| 4 | Water | 18.016 | 0.0000 | 0.0000 |
| Maverage (kg/kmol) |  | $\mathbf{5 8 . 0 8}$ | $\mathbf{5 8 . 0 8}$ | $\mathbf{1 5 9 . 3 8 7 4}$ |

- The physical properties of the fluids were obtained from the Aspen Hysys simulator database for each stream defined by its composition at the bottom of the tower and at the bubble and dew point temperature.

| CONDITIONS OF OPERATION |  |  |
| :---: | :---: | :---: |
| P $(\mathrm{kPa})$ | Pressure | 300 |
| $\mathrm{~g}(\mathrm{~m} / \mathrm{s} 2)$ | Gravity | 9.807 |


| PHYSICAL PROPERTIES OF FLUIDS |  |  |
| :---: | :---: | :---: |
| $\rho \mathrm{G}(\mathrm{kg} / \mathrm{m} 3)$ | Vapor Density | 5.742 |
| $\rho \mathrm{~L}(\mathrm{~kg} / \mathrm{m} 3)$ | Liquid Density | 710.7 |
| $\mu \mathrm{G}(\mathrm{kg} / \mathrm{m} . \mathrm{s})$ | Viscosity of steam | $8.06 \mathrm{E}-06$ |
| $\sigma(\mathrm{~N} / \mathrm{m})$ | Surface tension | 0.022 |
| $\mu \mathrm{~L}(\mathrm{~kg} / \mathrm{m} . \mathrm{s})$ | Viscosity liq alimentac | $2.10 \mathrm{E}-04$ |

- The following initial characteristics of the dish were taken from which the dimensions of the tower are calculated, as well as checking that there are no operational problems (flooding, whining, dragging, etc.).

| PLATE CHARACTERISTICS |  |  | OBSERVATIONS |
| :---: | :---: | :---: | :---: |
| $\mathrm{do}(\mathrm{m})$ | Diameter of the hole | 0.0045 | Tabla 6.2 - Item 2 |
| $\mathrm{p}^{\prime}(\mathrm{m})$ | Distance between holes | 0.0135 | $\mathrm{p}^{\prime}=(2.5-5)$. do |
| $\mathrm{t}(\mathrm{m})$ | Spacing Tower | 0.30 | Tabla 6.1 - Item 1 |

- The following preliminary flow calculations were performed:

$$
\begin{array}{ll}
\mathrm{G}^{\prime}=\mathrm{G} \cdot \overline{\mathrm{M}}_{\mathrm{avgG}}=2.572 \mathrm{~kg} / \mathrm{s} & \mathrm{Q}=\frac{\mathrm{G}^{\prime}}{\rho_{\mathrm{G}}}=0.45 \mathrm{~m}^{3} / \mathrm{s} \\
\mathrm{~L}^{\prime}=\mathrm{L} \cdot \overline{\mathrm{M}}_{\mathrm{avgL}}=2.821 \mathrm{~kg} / \mathrm{s} & \mathrm{q}=\frac{\mathrm{L}^{\prime}}{\rho_{\mathrm{L}}}=3.97 \times 10^{-3} \mathrm{~m}^{3} / \mathrm{s}
\end{array}
$$

## Diameter of Tower (T):

- An orifice diameter of $=4.5 \mathrm{~mm}$ has been taken on an equilateral triangle distribution with distances of $p^{\prime}=13.5 \mathrm{~mm}$ between the centers of the holes.

$$
\begin{equation*}
\frac{\mathrm{A}_{\mathrm{o}}}{\mathrm{~A}_{\mathrm{a}}}=0.907\left(\frac{\mathrm{~d}_{\mathrm{o}}}{\mathrm{p}^{\prime}}\right)^{2}=0.1008 \geq 0.1 \tag{Equation6.31}
\end{equation*}
$$

- From Table 6.2 - Item 1 we determine the ranges for the calculation of the constants $\alpha$ and $\beta$, considering a spacing of $t=0.30 \mathrm{~m}$.

$$
\begin{gathered}
\left.\frac{\mathrm{L}^{\prime}}{\mathrm{G}^{\prime}}\left(\frac{\rho_{\mathrm{G}}}{\rho_{\mathrm{L}}}\right)^{0.5}=0.0986 \in[0.01-0.1] \quad \quad \text { (Use values in } 0.1\right) \\
\alpha=0.0744 \mathrm{t}+0.01173=0.0341 \\
\beta=0.0304 \mathrm{t}+0.015=0.0241
\end{gathered}
$$

- For the calculation of the flooding constant (CF), values of (L/G ') ( $\rho \mathrm{G} / \rho \mathrm{L}$ ) 0.5 equal to 0.1 , since it is in the range of 0.01 to 0.1 .
$\mathrm{C}_{\mathrm{F}}=\left[\alpha \log \frac{1}{\left(\mathrm{~L}^{\prime} / \mathrm{G}^{\prime}\right)\left(\rho_{\mathrm{G}} / \rho_{\mathrm{L}}\right)^{0.5}}+\beta\right]\left(\frac{\sigma}{0.020}\right)^{0.2}=0.0592$

$$
\mathrm{V}_{\mathrm{F}}=\mathrm{C}_{\mathrm{F}}\left(\frac{\rho_{\mathrm{L}}-\rho_{\mathrm{G}}}{\rho_{\mathrm{G}}}\right)^{1 / 2}=0.66 \mathrm{~m} / \mathrm{s} \quad \text { (Equation 6.29) }
$$

- Using $80 \%$ of the flood velocity $\mathrm{V}=0.80$ 【V』_F $=0.52 \mathrm{~m} / \mathrm{s}$

$$
\mathrm{A}_{\mathrm{n}}=\frac{\mathrm{Q}}{\mathrm{~V}}=0.853 \mathrm{~m}^{2}
$$

- We choose a length of the spill $\mathrm{W}=0.80 \mathrm{~T}$, therefore from Table 6.1 - Item 4 $\mathrm{A}_{\mathrm{d}}=0.14145 \mathrm{At}$

$$
\begin{gathered}
A_{n}=A_{t}-A_{d}=A_{t}-0.14145 \mathrm{At}=(1-0.14145) \mathrm{A}_{\mathrm{t}} \\
A_{t}=\frac{A_{n}}{1-0.14145}=0.994 \mathrm{~m}^{2}
\end{gathered}
$$

$$
\mathrm{T}=\sqrt{\frac{4 \mathrm{~A}_{\mathrm{t}}}{\pi}}=1.125 \mathrm{~m}
$$

- We choose $\mathrm{T}^{\wedge}$ ' $=1.25 \mathrm{~m}$ as the diameter of the tower and proceed to correct the total area previously calculated:

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{t}}^{\prime}=\frac{\pi \mathrm{T}^{\prime 2}}{4}=1.227 \mathrm{~m}^{2} \\
& \mathrm{~W}=0.80 \mathrm{~T}^{\prime}=1.00 \mathrm{~m}
\end{aligned}
$$

$$
\mathrm{A}_{\mathrm{d}}=0.14145 \mathrm{~A}_{\mathrm{t}}^{\prime}=0.1736 \mathrm{~m}^{2}
$$

- From Table 6.2-Item 4 for a T $\wedge^{\wedge}=1.25$ $m$ is obtained $A_{a}=0.70 A_{t}{ }^{\prime}=$ $0.8590 \mathrm{~m}^{2}$, therefore checking the flow of the liquid:

$$
\begin{equation*}
\frac{\mathrm{q}}{\mathrm{~W}}=0.004 \frac{\mathrm{~m}^{3}}{\mathrm{~m} \cdot \mathrm{~s}}<0.015 \frac{\mathrm{~m}^{3}}{\mathrm{~m} \cdot \mathrm{~s}} \tag{О.K.}
\end{equation*}
$$



## Depth of Liquid (hw $+h_{1}$ ):

- We assume a height of the overflow (hW) of:

$$
\mathrm{h}_{\mathrm{W}}=50 \mathrm{~mm}
$$

- Iterate until $h_{1}=h_{1}{ }^{\prime}$, for $h_{1}=17.0 \mathrm{~mm}$ in Equation 6.34:

$$
\begin{gathered}
\left(\frac{W_{\text {eff }}}{W}\right)^{2}=\left(\frac{T}{W}\right)^{2}-\left\{\left[\left(\frac{T}{W}\right)^{2}-1\right]^{0.5}+\frac{2 \mathrm{~h}_{1}}{T} \frac{T}{W}\right\}^{2} \\
W_{\text {eff }}=0.9736 \mathrm{~m}
\end{gathered}
$$

- Replacing in Equation 6.33:

$$
\begin{gathered}
\mathrm{h}_{1}^{\prime}=0.666\left(\frac{\mathrm{q}}{\mathrm{~W}}\right)^{2 / 3}\left(\frac{\mathrm{~W}}{\mathrm{~W}_{\mathrm{eff}}}\right)^{2 / 3} \\
\mathrm{~h}_{1}^{\prime}=17.0 \mathrm{~mm}
\end{gathered}
$$

- Check the depth of the liquid:

$$
\begin{align*}
& \mathbf{h}_{\mathbf{W}}+\mathbf{h}_{\mathbf{1}}=\mathbf{6 7 . 0} \mathrm{mm}=\mathbf{0 . 0 6 7 0} \mathrm{m} \\
& 50 \mathrm{~mm}<\mathrm{h}_{\mathrm{W}}+\mathrm{h}_{1}<100 \mathrm{~mm} \tag{0.K.}
\end{align*}
$$

## Pressure Drop for Gas (hG):

- From Table 6.2 - Item 2 for a do $=4.5 \mathrm{~mm}$ and a stainless steel tower yields l / do $=0.43$, therefore the orifice coefficient (Co) will be:

$$
\begin{equation*}
\mathrm{C}_{\mathrm{o}}=1.09\left(\frac{\mathrm{~d}_{\mathrm{o}}}{\mathrm{l}}\right)^{0.25}=1.346 \tag{Equation6.37}
\end{equation*}
$$

- From Equation $6.31 \mathrm{~A}_{\mathrm{o}} / \mathrm{A}_{\mathrm{a}}=0.1008$, therefore $\mathrm{A}_{\mathrm{o}}=0.0866 \mathrm{~m}^{2}$ and the gas velocity through the holes will beV ${ }_{o}=Q / A_{o}=5.2 \mathrm{~m} / \mathrm{s}$.
- We calculate the Reynolds number (Re) and the Fanning friction factor (f), knowing that the absolute roughness for stainless steel is $\varepsilon=0.002 \mathrm{~mm}$.

$$
\begin{gathered}
\operatorname{Re}=\frac{\mathrm{d}_{\mathrm{o}} \cdot \mathrm{~V}_{\mathrm{o}} \cdot \rho_{\mathrm{G}}}{\mu_{\mathrm{G}}}=16586 \\
\mathrm{f}=\frac{1}{4}\left\{-1.8 \log \left[\frac{6.9}{\operatorname{Re}}+\left(\frac{\varepsilon / \mathrm{d}_{\mathrm{o}}}{3.7}\right)^{1.11}\right]\right\}^{-2}=0.00693
\end{gathered}
$$

- Calculate the pressure drop on dry plate (hD) with Equation 6.36:

$$
h_{D}=\frac{C_{o} V_{o}^{2} \rho_{G}}{2 g \rho_{L}}\left[0.40\left(1.25-\frac{A_{o}}{A_{n}}\right)+\frac{4 \text { lf }}{d_{o}}+\left(1-\frac{A_{o}}{A_{n}}\right)^{2}\right]=0.0190 \mathrm{~m}
$$

- We determine the average flow width $\mathrm{z}=(\mathrm{T}+\mathrm{W}) / 2=1.125 \mathrm{~m}$ and the speed in base of the active area $V_{a}=Q / A_{a}=0.521 \mathrm{~m} / \mathrm{s}$, therefore from Equation 6.38 we calculate the pressure drop of the liquid (hL).
$\mathrm{h}_{\mathrm{L}}=6.10 \times 10^{-3}+0.725 \mathrm{~h}_{\mathrm{W}}-0.238 \mathrm{~h}_{\mathrm{W}} \mathrm{V}_{\mathrm{a}} \rho_{\mathrm{G}}{ }^{0.5}+1.225 \frac{\mathrm{q}}{\mathrm{Z}}=0.0318 \mathrm{~m}$
- The residual pressure drop (hR) is calculated from Equation 6.42:

$$
\mathrm{h}_{\mathrm{R}}=\frac{6 \sigma \mathrm{~g}_{\mathrm{c}}}{\rho_{\mathrm{L}} \mathrm{~d}_{\mathrm{o}} \mathrm{~g}}=0.0042 \mathrm{~m}
$$

- Therefore the pressure drop for the gas (hG) will be:

$$
\begin{gathered}
h_{G}=h_{D}+h_{L}+h_{R}=0.0550 \mathrm{~m} \\
\Delta \mathbf{P}_{G}=\rho_{L} g h_{G}=356.1 \mathrm{~Pa}
\end{gathered}
$$

## Flood Verification

- Considering a 40 mm seal, we determine the smaller of two areas (Ada), the cross section of the landfill (Ad) or the free area between the landfill and the plate :

$$
\begin{gathered}
\mathrm{A}_{\mathrm{d}}=0.1736 \mathrm{~m}^{2} \quad \mathrm{~A}_{\text {free }}=\mathrm{W}\left(\mathrm{~h}_{\mathrm{W}}-0.04\right)=0.0100 \mathrm{~m}^{2} \\
\mathrm{~A}_{\mathrm{da}}=\mathrm{A}_{\text {free }}=0.0100 \mathrm{~m}^{2}
\end{gathered}
$$

- Calculate the pressure drop at the liquid inlet (h2) of Equation 6.43.

$$
\mathrm{h}_{2}=\frac{3}{2 \mathrm{~g}}\left(\frac{\mathrm{q}}{\mathrm{~A}_{\mathrm{da}}}\right)^{2}=0.0241 \mathrm{~m}
$$

- Therefore the difference in the level of the liquid inside and outside the landfill will be:

$$
\mathbf{h}_{3}=\mathbf{h}_{G}+h_{2}=0.0791 \mathrm{~m}
$$

- Finally verifying the flood:

$$
\begin{equation*}
\mathrm{h}_{\mathrm{W}}+\mathrm{h}_{1}+\mathrm{h}_{3}=0.1461 \mathrm{~m}<\frac{\mathrm{t}}{2} \tag{О.K.}
\end{equation*}
$$

## Whining Checking

- From Table 6.1 - Item 4 for a $\mathrm{W}=0.80 \mathrm{~T}$ is obtainedZ/ $2=0.1991 \mathrm{~T}$, thereforeZ $=0.498 \mathrm{~m}$.
- We calculate the speed ( $\mathrm{V}_{\text {ow }}$ ) of Equation 6.46.

$$
\mathrm{V}_{\mathrm{oW}}=\frac{0.0229 \sigma \mathrm{~g}_{\mathrm{c}}}{\mu_{\mathrm{G}}}\left(\frac{\mu_{\mathrm{G}}{ }^{2}}{\sigma \mathrm{~g}_{\mathrm{c}} \rho_{\mathrm{G}} \mathrm{~d}_{\mathrm{o}}} \frac{\rho_{\mathrm{L}}}{\rho_{\mathrm{G}}}\right)^{0.379}\left(\frac{\mathrm{l}}{\mathrm{~d}_{\mathrm{o}}}\right)^{0.293}\left(\frac{2 \mathrm{~A}_{\mathrm{a}} \mathrm{~d}_{\mathrm{o}}}{\sqrt{3} \mathrm{p}^{\prime 3}}\right)^{2.8 /\left(\mathrm{Z} / \mathrm{d}_{\mathrm{o}}\right)^{0.724}}=1.42 \mathrm{~m} / \mathrm{s}
$$

- whining check:

$$
\begin{equation*}
\mathrm{V}_{\mathrm{o}}=3.08 \mathrm{~m} / \mathrm{s}>\mathrm{V}_{\mathrm{oW}} \tag{О.К.}
\end{equation*}
$$

## Liquid Crawl Verification

- To determine the drag (E) we will use Figure 6.17, for which we enter by the abscissa with the value of:

$$
\frac{\mathrm{L}^{\prime}}{\mathrm{G}^{\prime}}\left(\frac{\rho_{\mathrm{G}}}{\rho_{\mathrm{L}}}\right)^{0.5}=0.10
$$

- Up to the curve V / (V_F = 0.80), therefore of the figure: $\quad \mathbf{E}=\mathbf{0 . 0 3}$
- The drag is so small that it does not significantly modify the hydraulics of the plate.


## Plate Global efficiency

- To determine the overall efficiency of the plate (Eo) we will use Figure 6.25, for which we enter by the abscissa with the value of:

$$
\alpha_{A V} \mu_{L}=3.2 \times 10^{-4}
$$

- Therefore of the figure: $\quad$ Eo $=\mathbf{0 . 6 5}$

| TOWER DISTILLATION IPA |  |  |
| :---: | :--- | :---: |
| $\mathrm{T}(\mathrm{m})$ | Diameter of the tower | 1.25 |
| Nr | Number of actual dishes | 58 |
| Nf | Food dish | 14 |
| $\mathrm{H}(\mathrm{m})$ | Tower height | 17.10 |
| $\mathrm{e} \mathrm{(mm)}$ | Wall Thickness | 7 |
| $\mathrm{M} \mathrm{(kg)}$ | Mass of cover | 3775 |

## ANNEX N ${ }^{1} 10$

## Acetone reflux pumps

From the following data obtained from the material balance.

| Current | Recycling of acetone |  |  |
| :--- | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 45.780 | 45.780 |  |
| Pressure (kPa) | 260.00 | 280.00 |  |
| Fraction of vapor | 0 | 0 |  |
| Molar flow (kmol / hr) | 117.19 | 117.19 |  |
| Components |  |  |  |
| Molar flow (kmol / hr) |  |  |  |
| HYDROGEN | 0 | 0 |  |
| ACETONE | 117.072 | 117.072 |  |
| IPA | 0.117 | 0.117 |  |
| H20 | 0 | 0 |  |
| Mass flow (kg / h) | 6806.56 | 6806.56 |  |
| Density (kg / m3) | 756.6000 | 756.6000 |  |
| Flow rate (m3 / s) | 0.00250 | 0.00250 |  |

Performing energy balance at pump inlet and outlet:

$$
\begin{gathered}
\mathrm{E}_{1}+\mathrm{H}_{\mathrm{B}}=\mathrm{E}_{2} \\
\frac{\mathrm{P}_{1}}{\gamma}+\mathrm{Z}_{1}+\frac{\overline{\mathrm{V}}_{1}}{2 g}+\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}+\mathrm{Z}_{2}+\frac{\overline{\mathrm{V}}_{2}}{2 g}
\end{gathered}
$$

$$
\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}-\frac{\mathrm{P}_{1}}{\gamma}=\frac{300.2 \mathrm{kPa}}{9.81 \mathrm{~N} \times 777.6 \frac{\mathrm{~kg}}{\mathrm{~m}^{3}}}=39.354 \mathrm{~m}
$$

For the design, the data of the impeller diameter, roughness and rotational speed are considered.

| Design data |  |
| :--- | :---: |
| Impeller Diameter Da (m) | 0.371 |
| Pump head H (m) | 2.6946 |
| Rotational speed n (1 / sec) | 1.4 |
| Power of pump P (W) | 49.979145 |
| Discharge flow Q (m3 / s) | 0.00250 |
| Viscosity (Kg / m.s) | 0.001 |
| Density (Kg / m3) | 756.6000 |
| Roughness $\square$ (m) | 0.0001 |
| Coefficient of capacity Cq | 0.0350 |
| Coefficient of head Ch | 97.985 |
| Power factor CP | 3.4251 |

## ANNEX N ${ }^{\circ} 11$

## Acetone column bottom reboiler

From the following data:

| Current | V | B | $\mathrm{V}+\mathrm{B}$ | V 1 | V 2 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 107.00 | 107.00 | 106.5 | 165.00 | 165.00 |
| Temperature (K) | 380.15 | 380.15 | 379.65 | 438.15 | 438.15 |
| Pressure (kPa) | 300.00 | 300.00 | 300 | 700.00 | 700.00 |
| Fraction of vapor | 1.00 | 0.00 | 0.00 | 1.00 | 0.00 |
| Molar flow (kmol / hr) | 159.5026 | 33.2199 | 192.7225 | 248.7051 | 248.7051 |
| Components |  |  |  |  |  |
| Molar flow (kmol / hr) |  |  |  |  |  |
| H20 | 61.80 | 26.03 | 87.83 | 248.7051 | 248.7051 |
| IPA | 53.24 | 5.05 | 58.29 | 0.0000 | 0.0000 |
| HYDROGEN | 0.00 | 0.00 | 0.00 | 0.0000 | 0.0000 |
| ACETONE | 44.46 | 2.14 | 46.61 | 0.0000 | 0.0000 |
| Molar fractions |  |  |  |  |  |
| H20 | 0.3875 | 0.7834 | 0.4557 | 1.000 | 1.000 |
| IPA | 0.3338 | 0.1521 | 0.3024 | 0.000 | 0.000 |
| HYDROGEN | 0.0000 | 0.0000 | 0.0000 | 0.000 | 0.000 |
| ACETONE | 0.2788 | 0.0645 | 0.2418 | 0.000 | 0.000 |
| Mass flow (kg / h) | 6895.0937 | 896.8992 | 7791.9928 | 4476.6925 | 4476.6925 |

## A. Heat balance.

For design considerations for a vaporizer, only $80 \%$ of the inlet flow to the schematic can be vaporized:

$$
\text { Vapor } \text { Ratio }=\frac{159.5026}{192.7225}=0.8276
$$

The outline of the team is as follows:


For the determination of the overall transfer coefficient, we have the following calculation sequence:

## B. Determination of water vapor flow.

The heat required for the vaporization of the inlet flow is:

$$
Q_{p}=9280000 \frac{\mathrm{~kJ}}{\mathrm{~h}}
$$

The required steam flow rate $\left(165^{\circ} \mathrm{C}\right.$ and 700 kPa$)$ :

$$
\dot{m}_{\text {water vapor }}=\frac{9280000}{2072.959}=4476.6925 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

## C. Determination of LMTD.

Since the heating medium (water vapor) has a constant temperature profile, it is feasible to calculate the LMTD as follows:

$$
L M T D=\frac{(438.15 K-380.15 K)-(438.15 K-379.65 K)}{\ln \left(\frac{(438.15 K-380.15 K)}{(438.15 K-379.65 K)}\right)}=58.2496 K
$$

## D. Determination of the overall coefficient.

- Coefficient of heat transfer per armor - ho.

Outer tube diameter: $\quad D_{o}=0.0191 \mathrm{~m}^{2}$
Separation between Pitch and Do: $\quad C^{\prime}=0.0064 \mathrm{~m}^{2}$
Armor Diameter: $\quad D_{c}=0.3366 m(13.25$ pulg. $)$
Space between baffles: $B=0.0673 \mathrm{~m}$
Armored area: $\quad$ As $=\frac{0.0048 \times 0.3366 \times 0.0673}{0.0254}=0.0057 \mathrm{~m}^{2}$
Mixing Flux: $\quad G_{t}=\frac{7791.7273 \frac{\mathrm{~kg}}{\mathrm{~h} \cdot \mathrm{~m} 2}}{0.0057 \mathrm{~m}^{2} \times 3600}=379.189 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}}$
To $\mathrm{Tm}=107^{\circ} \mathrm{C}$, properties of the fluid are:

$$
\mu=0.00028 \text { Pa.s } \quad C p=3.7481 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \quad k=0.5074 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
$$

The equivalent diameter of the armor:

$$
D e=4 x\left(\frac{0.8661 x\left(\left(\left(\frac{0.0254}{0.0254}\right)\right)^{2}-\frac{\pi}{4} x\left(\frac{0.0191}{0.0254}\right)^{2}\right)}{\pi x\left(\frac{0.0191}{0.0254}\right)}\right) x 0.0254=0.01815 \mathrm{~m}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{379.189 \times 0.01815 \mathrm{~m}}{0.00028 \text { Pa. } \mathrm{s}}=24953.01
$$

From Fig. 24 of the book "Heat Transfer Processes" by Donald Kern, Factor Jh is extracted for the given Reynolds number:

$J h=90.00$

So:

$$
H o=\operatorname{Jh} \times \frac{k}{D e} \times\left(\frac{C \mu}{k}\right)^{1 / 3}=3190.9452 \frac{W}{m^{2} . K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

For purposes of calculation, the hi recommended by Donald Kern for steam flow:

$$
h_{i}=8000 \frac{W}{m^{2} \cdot K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{8000 \times 3190.9452}{8000+3190.9452}=2281.0908 \frac{W}{m^{2} \cdot K}
$$

Area required for heating:

$$
A 1=\frac{\frac{9280000}{3.6 \times 58.2496}}{2281.0908}=19.4004 \mathrm{~m}^{2}
$$

## E. Checking the maximum flow of heat.

- Total clean area Ac:

$$
A c=A 1=19.4004 \mathrm{~m}^{2}
$$

- Total design coefficient:

Total area $=N t \times L T \times R T=86 \times 5 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m}=$ $25.7279 \mathrm{~m}^{2}$ :

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Tube surface ratio per linear foot (BWG 16)

$$
U=\frac{\frac{9280000}{3.6 \times 58.2496}}{25.7279}=172.0078 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot K}
$$

## ANNEX N ${ }^{\circ} 12$

## IPA Distillation Tower

- Knowing the composition of the feed stream (F), we define the IPA as light key (LK) and water as heavy key (HK), whose compositions depend on the separation to be made. The balance of matter is obtained by obtaining:

| Molar composition and molar fluxes of DISTILLATE and FUNDS |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{D}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{B}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{x D}$ | $\mathbf{x B}$ |
| Acetone | 2.1429 | 2.1429 | 0.0000 | 0.1996 | 0.0000 |
| IPA | 5.0518 | 5.0496 | 0.0022 | 0.4704 | 0.0001 |
| Water | 26.0252 | 3.5426 | 22.4826 | 0.3300 | 0.9999 |
| TOTAL | $\mathbf{3 3 . 2 1 9 9}$ | $\mathbf{1 0 . 7 3 5 0}$ | $\mathbf{2 2 . 4 8 4 9}$ | $\mathbf{1 . 0 0 0 0}$ | $\mathbf{1 . 0 0 0 0}$ |

- We determine the temperature of dew and bubble of all the currents of the tower, since they will be necessary in the design of the condenser and reboiler. The procedure to be performed is the same as the one performed in the previous design.

| Bubble Point supply |  |  | Temperature ( ${ }^{\circ} \mathbf{C}$ ) |  | Pressure (kPa) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| It is iterated until: | $\sum(\mathbf{y i} / \boldsymbol{\alpha})-\mathbf{K r}$ | 0.0000 | 125.90 |  | 300 |  |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{y i}$ | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\mathbf{y i} / \boldsymbol{\alpha}$ | $\mathbf{x i}$ |
| Hydrogen | 2.1429 | 0.0645 | 18.0659 | 22.70 | 0.0028 | 0.0036 |
| Acetone | 5.0518 | 0.1521 | 12.6749 | 15.93 | 0.0095 | 0.0120 |
| IPA | 26.0252 | 0.7834 | 0.7958 | 1.00 | 0.7834 | 0.9844 |
| Water | $\mathbf{3 3 . 2 1 9 9}$ | $\mathbf{1 . 0 0 0 0}$ |  |  | $\mathbf{0 . 7 9 5 8}$ | $\mathbf{1 . 0 0 0 0}$ |


| Dew Point supply |  |  | Temperatura $\left({ }^{\circ} \mathbf{C}\right)$ |  | Presión (kPa) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| It is iterated until: | $\mathbf{1} / \sum \boldsymbol{\alpha} . \mathbf{x i}-\mathbf{K r}$ | 0.0000 | 107.00 |  | 300 |  |
| Components | $\mathbf{F}(\mathbf{k m o l} / \mathbf{h})$ | $\mathbf{x i}$ | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\boldsymbol{\alpha} . \mathbf{x i}$ | $\mathbf{y i}$ |
| Hydrogen | 2.1429 | 0.0645 | 4.3216 | 8.74 | 0.5636 | 0.2788 |
| Acetone | 5.0518 | 0.1521 | 2.1947 | 4.44 | 0.6748 | 0.3338 |
| IPA | 26.0252 | 0.7834 | 0.4946 | 1.00 | 0.7834 | 0.3875 |
| Water | $\mathbf{3 3 . 2 1 9 9}$ | $\mathbf{1 . 0 0 0 0}$ |  |  | $\mathbf{2 . 0 2 1 9}$ | $\mathbf{1 . 0 0 0 0}$ |

- Proceeding in the same way with the other currents you get:

| CURRENT | $\mathbf{P ( k P a})$ | $\mathbf{T}_{\text {burb }}\left({ }^{\circ} \mathbf{C}\right)$ | $\mathbf{T}_{\text {rocio }}\left({ }^{\circ} \mathbf{C}\right)$ |
| :---: | :---: | :---: | :---: |
| Supply | 300 | 107.00 | 125.90 |


| Distilled | 250 | 96.45 | 101.00 |
| :---: | :---: | :---: | :---: |
| Money | 270 | 130.00 | 129.90 |

- We applied the ShortCut Method to determine the number of ideal dishes (Nid) for a reflux ( $\mathrm{R}=1.25 \mathrm{Rm}$ ), the feed plate ( Nf ), the minimum number of plates (Nm) and minimum reflux tower. The first step is to verify if the components are distributed as initially assumed, for this the Shiras equation is applied:

$$
\frac{\mathrm{x}_{\mathrm{JD}} \mathrm{D}}{\mathrm{z}_{\mathrm{JF}} \mathrm{~F}}=\left(\frac{\alpha_{\mathrm{J}}-1}{\alpha_{\mathrm{lk}}-1}\right) \frac{\mathrm{x}_{\mathrm{lkD}} \mathrm{D}}{\mathrm{z}_{\mathrm{lkF}} \mathrm{~F}}+\left(\frac{\alpha_{\mathrm{lk}}-\alpha_{\mathrm{J}}}{\alpha_{\mathrm{lk}}-1}\right) \frac{\mathrm{x}_{\mathrm{hkD}} \mathrm{D}}{\mathrm{z}_{\mathrm{hkF}} \mathrm{~F}}
$$

- A component will be distributed if $0 \leq \mathrm{xJDD} / \mathrm{zJFF} \leq 1$, therefore:

| Key component check |  |  |  |  | Tavg ( ${ }^{\circ} \mathrm{C}$ ) |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| We assume Ki $=[(\mathrm{Ki}) \text { top.(Ki)bot }]^{\wedge} 0.5$ |  |  |  |  |  |  |
| Components | F (kmol/h) | D (kmol/h) | Ki | $\alpha=\mathbf{K i} / \mathbf{K r}$ | XJd.D/Z ${ }_{\text {JF }}$.F | ¿Distribuye? |
| Acetone | 2.1429 | 2.1429 | 7.96 | 9.35 | 1.9319 | No |
| IPA | 5.0518 | 5.0496 | 4.27 | 5.01 | 0.9996 | Si |
| Water | 26.0252 | 3.5426 | 0.85 | 1.00 | 0.1361 | Si |
| TOTAL | 33.2199 | 10.7350 |  |  |  |  |


| Conditions |  |
| :---: | :---: |
| Temperature ( $\left.{ }^{\circ} \mathbf{C}\right)$ | 107.00 |
| HF (kJ/kmol) | $-2.795 \mathrm{E}+05$ |
| $\mathbf{H G}(\mathbf{k J} / \mathbf{k m o l})$ | $-2.398 \mathrm{E}+05$ |
| $\mathbf{H L}(\mathbf{k J} / \mathbf{k m o l})$ | $-2.795 \mathrm{E}+05$ | - We calculate the minimum backflow (Rm) using the Underwood method, for this it is necessary to determine the thermal condition of the feed mixture ("q") and the value of $\varphi$ is obtained by iterating for values among the $\alpha$ components distributed.

$$
\begin{gathered}
\mathrm{q}=\frac{\mathrm{H}_{\mathrm{G}}-\mathrm{H}_{\mathrm{F}}}{\mathrm{H}_{\mathrm{G}}-\mathrm{H}_{\mathrm{L}}} \\
\mathrm{q}=\frac{-2.398-(-2.795)}{-2.398-(-2.795)}=1.000
\end{gathered}
$$

- The values of q and $\varphi$ are replaced in the following equations:

| Calculation of the minimum reflux (Rm) |  |  |  | $\boldsymbol{\varphi}=$ |
| :---: | :---: | :---: | :---: | :---: |
| To supply $\mathbf{q}=$ | 1.0000 | $\mathbf{1 - q - \sum \boldsymbol { \alpha } \mathbf { z F } / ( \boldsymbol { \alpha } - \boldsymbol { \varphi } ) =}$ |  | 0.7971 |
| Components | $\mathbf{K i}$ | $\boldsymbol{\alpha}=\mathbf{K i} / \mathbf{K r}$ | $\boldsymbol{\alpha} \cdot \mathbf{z F} /(\boldsymbol{\alpha}-\boldsymbol{\varphi})$ | $[\boldsymbol{\alpha} . \mathbf{x D} /(\boldsymbol{\alpha}-\boldsymbol{\varphi})] \cdot \mathbf{D}$ |
| Acetone | 7.96 | 9.35 | 0.0920 | 3.0577 |


| IPA | 4.27 | 5.01 | 0.3439 | 11.4194 |
| :---: | :---: | :---: | :---: | :---: |
| Water | 0.85 | 1.00 | -0.4359 | -1.9713 |
| TOTAL |  |  | $\mathbf{0 . 0 0 0 0}$ | $\mathbf{1 2 . 5 0 5 7}$ |

$$
\sum \frac{\alpha_{\mathrm{J}} \mathrm{Z}_{\mathrm{JF}} \mathrm{~F}}{\alpha_{\mathrm{J}}-\varphi}=\mathrm{F}(1-\mathrm{q}) \quad \sum \frac{\alpha_{\mathrm{J}} \mathrm{X}_{\mathrm{JD}} \mathrm{D}}{\alpha_{\mathrm{J}}-\varphi}=\mathrm{D}\left(\mathrm{R}_{\mathrm{m}}+1\right)
$$

| $\mathbf{D}(\mathbf{R m}+\mathbf{1})$ | 12.5057 |
| :---: | :---: |
| $\mathbf{R m}$ | $\mathbf{0 . 1 6 4 9}$ |

- The minimum number of plates $(\mathrm{Nm})$ is determined by the equation of Fenske:

$$
\begin{aligned}
& \mathrm{N}_{\mathrm{m}}=\frac{\log \left[\left(\frac{\mathrm{x}_{\mathrm{lk}}}{\mathrm{x}_{\mathrm{hk}}}\right)_{\mathrm{D}}\left(\frac{\mathrm{x}_{\mathrm{hk}}}{\mathrm{x}_{\mathrm{lk}}}\right)_{\mathrm{B}}\right]}{\log \left(\alpha_{\mathrm{lkAV}}\right)}-1 \\
& \alpha_{\mathrm{lk} A V}=\sqrt{\left(\frac{\mathrm{K}_{\mathrm{lk}}}{\mathrm{~K}_{\mathrm{hk}}}\right)_{\mathrm{D}}\left(\frac{\mathrm{~K}_{\mathrm{lk}}}{\mathrm{~K}_{\mathrm{hk}}}\right)_{\mathrm{B}}}
\end{aligned}
$$

- The number of theoretical plates $(\mathrm{N})$ for a given reflux ratio ( $\mathrm{R}=1.25 \mathrm{Rm}$ ) is determined by the correlation of Gilliland:

$$
\begin{aligned}
& X=\frac{R-R_{m}}{R+1} \\
& \frac{N-N_{m}}{N+1}=1-e^{\left[\frac{1+54.4 X}{11+117.2 X}\left(\frac{X-1}{X^{0.5}}\right)\right]}
\end{aligned}
$$

| NR/NS | 0.06 |
| :---: | :--- |
| NR | $\mathbf{0 . 8 2}$ |



Kirkbride equation gives an approximate location of the supply plate:

$$
\frac{\mathrm{N}_{\mathrm{R}}}{\mathrm{~N}_{\mathrm{S}}}=\left[\left(\frac{\mathrm{z}_{\mathrm{hkF}}}{\mathrm{z}_{\mathrm{lkF}}}\right)\left(\frac{\mathrm{x}_{\mathrm{lkB}}}{\mathrm{x}_{\mathrm{hkD}}}\right)\left(\frac{\mathrm{B}}{\mathrm{D}}\right)\right]^{0.206}
$$

- For the design of perforated plates of cross-flow of a single step was taken into consideration the maximum flows of both liquid and vapor in the tower, the procedure carried out is the same as the one performed for the previous design.
- The physical properties of the fluids were obtained from the Aspen Hysys

|  | COMPOSITION AND MOLAR FLUX OF LIQUID |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | L (kmol/s) |  | 0.0098 |  |  |  |
|  |  |  | PHYSICAL PROPERTIES OF FLUIDS |  |  |  |
| CONDITIONS OF OPERATION |  |  |  |  |  | 3.952 |
| P (kPa) | 1 PressureAcetone270.0 |  |  | $58.088 \mathrm{D}(\mathrm{kg} / \mathrm{mb})$. 0729 |  | 798.8 |
| $\mathrm{g}(\mathrm{m} / \mathrm{s} 2)$ | 2 Gravity IPA 9.807 |  |  | $60.480(\mathrm{~kg} / \mathrm{m} .8) 1$ 120 | Yisc6s0日330f steam | 8.65E-06 |
|  | 3 | Water |  | 18.016 ( $\mathrm{N} / \mathrm{m}$ ) $0.7 \$ 51$ | sulla.cefefnion | 0.047 |
|  |  | (kg/kmol) |  | 28.142 (kg/m.1)0000V | sc35id43136 supply | $2.95 \mathrm{E}-04$ |

simulator database for each stream
defined by its composition at the bottom of the tower and at the bubble and dew point temperature.

- The following initial characteristics of the dish were taken from which the dimensions of the tower are calculated, as well as checking that there are no operational problems (flooding, whining, dragging, etc.).

| PLATE CHARACTERISTICS |  |  | OBSERVATIONS |
| :---: | :---: | :---: | :---: |
| do $(\mathrm{m})$ | Diameter of the hole | 0.0045 | Tabla 6.2 - IItem 2 |
| $\mathrm{p}^{\prime}(\mathrm{m})$ | Distance between holes | 0.015 | $\mathrm{p}^{\prime}=(2.5-5)$. do |
| $\mathrm{t}(\mathrm{m})$ | Spacing Tower | 0.25 | Tabla 6.1 - Îtem 1 |

- The following preliminary calculations of flows:

$$
\begin{array}{ll}
\mathrm{G}^{\prime}=\mathrm{G} \cdot \overline{\mathrm{M}}_{\mathrm{avgG}}=0.165 \mathrm{~kg} / \mathrm{s} & \mathrm{Q}=\frac{\mathrm{G}^{\prime}}{\rho_{\mathrm{G}}}=0.04 \mathrm{~m}^{3} / \mathrm{s} \\
\mathrm{~L}^{\prime}=\mathrm{L} \cdot \overline{\mathrm{M}}_{\mathrm{avgL}}=0.277 \mathrm{~kg} / \mathrm{s} & \mathrm{q}=\frac{\mathrm{L}^{\prime}}{\rho_{\mathrm{L}}}=3.47 \times 10^{-4} \mathrm{~m}^{3} / \mathrm{s}
\end{array}
$$

## Diameter of Tower (T):

- An orifice diameter of $=4.5 \mathrm{~mm}$ has been taken on an equilateral triangle distribution with distances p ' $=15 \mathrm{~mm}$ between the centers of the holes.

$$
\frac{A_{o}}{A_{a}}=0.907\left(\frac{d_{o}}{p^{\prime}}\right)^{2}=0.0816<0.1 \quad \text { (Ecuación 6.31) }
$$

- From Table 6.2 - Item 1 we determine the ranges for the calculation of the constants $\alpha$ and $\beta$, considering a spacing of $t=0.25 \mathrm{~m}$.

$$
\begin{gathered}
\frac{\mathrm{L}^{\prime}}{\mathrm{G}^{\prime}}\left(\frac{\rho_{\mathrm{G}}}{\rho_{\mathrm{L}}}\right)^{0.5}=0.1184 \\
\alpha=0.0744 \mathrm{t}+0.01173=0.0275 \\
\beta=0.0304 \mathrm{t}+0.015=0.0205
\end{gathered}
$$

- Calculation of the flood constant $\left(\mathrm{C}_{\mathrm{F}}\right)$ :

$$
\begin{gather*}
\mathrm{C}_{\mathrm{F}}=\left[\alpha \log \frac{1}{\left(\mathrm{~L}^{\prime} / \mathrm{G}^{\prime}\right)\left(\rho_{\mathrm{G}} / \rho_{\mathrm{L}}\right)^{0.5}}+\beta\right]\left(\frac{\sigma}{0.020}\right)^{0.2}=0.0547  \tag{Equation6.30}\\
\mathrm{~V}_{\mathrm{F}}=\mathrm{C}_{\mathrm{F}}\left(\frac{\rho_{\mathrm{L}}-\rho_{\mathrm{G}}}{\rho_{\mathrm{G}}}\right)^{1 / 2}=0.78 \mathrm{~m} / \mathrm{s}
\end{gather*}
$$

- Using $70 \%$ of the flood speedV $=0.70 \mathrm{~V}_{\mathrm{F}}=0.54 \mathrm{~m} / \mathrm{s}$

$$
A_{n}=\frac{Q}{V}=0.077 \mathrm{~m}^{2}
$$

- We choose a length of the spill $\mathrm{W}=0.80 \mathrm{~T}$, therefore from Table 6.1 - Item 4 $\mathrm{A}_{\mathrm{d}}=0.14145 \mathrm{At}$

$$
A_{n}=A_{t}-A_{d}=A_{t}-0.14145 A t=(1-0.14145) A_{t}
$$

$$
A_{t}=\frac{A_{n}}{1-0.14145}=0.089 \mathrm{~m}^{2}
$$

$$
\mathrm{T}=\sqrt{\frac{4 \mathrm{~A}_{\mathrm{t}}}{\pi}}=0.338 \mathrm{~m}
$$

- We choose $\mathrm{T}^{\wedge}{ }^{\prime}=0.50 \mathrm{~m}$ as the diameter of the tower and proceed to correct the total area previously calculated:

$$
\begin{gathered}
\mathrm{A}_{\mathrm{t}}^{\prime}=\frac{\pi \mathrm{T}^{\prime 2}}{4}=0.196 \mathrm{~m}^{2} \\
\mathrm{~W}=0.80 \mathrm{~T}^{\prime}=0.4 \mathrm{~m} \\
\mathrm{~A}_{\mathrm{d}}=0.14145 \mathrm{~A}_{\mathrm{t}}^{\prime}=0.0278 \mathrm{~m}^{2}
\end{gathered}
$$

- From Table 6.2 - Item 4 for a $T^{\prime}=0.50 \mathrm{~m}$ we obtain $\mathrm{A}_{\mathrm{a}}=0.51 \mathrm{~A}_{\mathrm{t}}{ }^{\prime}=$ $0.0997 \mathrm{~m}^{2}$, therefore verifying the flow of the liquid:

$$
\begin{equation*}
\frac{\mathrm{q}}{\mathrm{~W}}=0.0009 \frac{\mathrm{~m}^{3}}{\mathrm{~m} \cdot \mathrm{~s}}<0.015 \frac{\mathrm{~m}^{3}}{\mathrm{~m} \cdot \mathrm{~s}} \tag{О.К.}
\end{equation*}
$$

## Depth of Liquid (hw $+h_{1}$ ):

- We assume a height of the overflow (hW) of:

$$
\mathrm{h}_{\mathrm{W}}=50 \mathrm{~mm}
$$

- Iterate until $\mathrm{h}_{1}=\mathrm{h}_{1}{ }^{\prime}$, para $\mathrm{h}_{1}=6.2 \mathrm{~mm}$ in Equation 6.34:

$$
\begin{gathered}
\left(\frac{W_{\text {eff }}}{W}\right)^{2}=\left(\frac{T}{W}\right)^{2}-\left\{\left[\left(\frac{T}{W}\right)^{2}-1\right]^{0.5}+\frac{2 h_{1}}{T} \frac{T}{W}\right\}^{2} \\
W_{\text {eff }}=0.3904 \mathrm{~m}
\end{gathered}
$$

- Replacing in Equation 6.33:


$$
\begin{gathered}
\mathrm{h}_{1}^{\prime}=0.666\left(\frac{\mathrm{q}}{\mathrm{~W}}\right)^{2 / 3}\left(\frac{\mathrm{~W}}{\mathrm{~W}_{\text {eff }}}\right)^{2 / 3} \\
\mathrm{~h}_{1}^{\prime}=6.2 \mathrm{~mm}
\end{gathered}
$$

- Check the depth of the liquid:

$$
\begin{align*}
& \mathbf{h}_{\mathbf{W}}+\mathbf{h}_{\mathbf{1}}=\mathbf{5 6 . 2} \mathbf{~ m m}=\mathbf{0 . 0 5 6 2} \mathbf{~ m} \\
& 50 \mathrm{~mm}<\mathrm{h}_{\mathrm{W}}+\mathrm{h}_{1}<100 \mathrm{~mm} \tag{0.K.}
\end{align*}
$$

## Pressure Drop for Gas (hG):

- From Table 6.2 - Item 2 for a do $=4.5 \mathrm{~mm}$ and a stainless steel tower yields $\mathrm{l} / \mathrm{do}$ $=0.43$, therefore the orifice coefficient (Co) will be:

$$
\begin{equation*}
C_{o}=1.09\left(\frac{d_{o}}{\mathrm{l}}\right)^{0.25}=1.346 \tag{Equation6.37}
\end{equation*}
$$

- From equation $6.31 \mathrm{~A}_{o} / \mathrm{A}_{\mathrm{a}}=0.0816$, therefore $\mathrm{A}_{\mathrm{o}}=0.0081 \mathrm{~m}^{2}$ and the gas velocity through the holes will beV ${ }_{o}=Q / A_{o}=5.1 \mathrm{~m} / \mathrm{s}$.
- We calculate the Reynolds number (Re) and the Fanning friction factor (f), knowing that the absolute roughness for stainless steel is $\varepsilon=0.002 \mathrm{~mm}$.

$$
\begin{gathered}
\operatorname{Re}=\frac{\mathrm{d}_{\mathrm{o}} \cdot \mathrm{~V}_{\mathrm{o}} \cdot \rho_{\mathrm{G}}}{\mu_{\mathrm{G}}}=10530 \\
\mathrm{f}=\frac{1}{4}\left\{-1.8 \log \left[\frac{6.9}{\operatorname{Re}}+\left(\frac{\varepsilon / \mathrm{d}_{\mathrm{o}}}{3.7}\right)^{1.11}\right]\right\}^{-2}=0.00775
\end{gathered}
$$

- Calculate the pressure drop on dry plate (hD) with Equation 6.36:

$$
\mathrm{h}_{\mathrm{D}}=\frac{\mathrm{C}_{\mathrm{o}} \mathrm{~V}_{\mathrm{o}}^{2} \rho_{\mathrm{G}}}{2 \mathrm{~g} \rho_{\mathrm{L}}}\left[0.40\left(1.25-\frac{\mathrm{A}_{\mathrm{o}}}{\mathrm{~A}_{\mathrm{n}}}\right)+\frac{4 \mathrm{lf}}{\mathrm{~d}_{\mathrm{o}}}+\left(1-\frac{\mathrm{A}_{\mathrm{o}}}{\mathrm{~A}_{\mathrm{n}}}\right)^{2}\right]=0.0113 \mathrm{~m}
$$

- We determine the average flow width $\mathrm{z}=((\mathrm{T}+\mathrm{W})) / 2=0.450 \mathrm{~m}$ and the velocity based on the active area $\mathrm{V} \_\mathrm{a}=\mathrm{Q} / \mathrm{A} \_\mathrm{a}=0.418 \mathrm{~m} / \mathrm{s}$, therefore from Equation 6.38 we calculate the pressure drop of the liquid (hL).

$$
\mathrm{h}_{\mathrm{L}}=6.10 \times 10^{-3}+0.725 \mathrm{~h}_{\mathrm{W}}-0.238 \mathrm{~h}_{\mathrm{W}} \mathrm{~V}_{\mathrm{a}} \rho_{\mathrm{G}}^{0.5}+1.225 \frac{\mathrm{q}}{\mathrm{z}}=0.0334 \mathrm{~m}
$$

- The residual pressure drop (hR) is calculated from Equation 6.42:

$$
\mathrm{h}_{\mathrm{R}}=\frac{6 \sigma \mathrm{~g}_{\mathrm{c}}}{\rho_{\mathrm{L}} \mathrm{~d}_{\mathrm{o}} \mathrm{~g}}=0.0080 \mathrm{~m}
$$

- Therefore the pressure drop for the gas ( hG ) will be:

$$
\begin{gathered}
h_{G}=h_{D}+h_{L}+h_{R}=0.0527 \mathrm{~m} \\
\Delta \mathbf{P}_{G}=\rho_{L} g h_{G}=413.2 \mathrm{~Pa}
\end{gathered}
$$

## Flood Verification

- Considering a 30 mm seal, we determine the smaller of two areas (Ada), the crosssection of the landfill (Ad) or the free area between the landfill and the plate (Afree):

$$
\begin{gathered}
\mathrm{A}_{\mathrm{d}}=0.0278 \mathrm{~m}^{2} \quad \mathrm{~A}_{\text {free }}=\mathrm{W}\left(\mathrm{~h}_{\mathrm{W}}-0.03\right)=0.0080 \mathrm{~m}^{2} \\
\mathrm{~A}_{\mathrm{da}}=\mathrm{A}_{\text {free }}=0.0080 \mathrm{~m}^{2}
\end{gathered}
$$

- Calculate the pressure drop at the liquid inlet (h2) of Equation 6.43.

$$
\mathrm{h}_{2}=\frac{3}{2 \mathrm{~g}}\left(\frac{\mathrm{q}}{\mathrm{~A}_{\mathrm{da}}}\right)^{2}=0.0003 \mathrm{~m}
$$

- Therefore the difference in the level of the liquid inside and outside the landfill will be:

$$
\mathbf{h}_{3}=\mathbf{h}_{\mathrm{G}}+\mathbf{h}_{2}=0.0530 \mathrm{~m}
$$

- Finally verifying the flood:

$$
\begin{equation*}
\mathrm{h}_{\mathrm{W}}+\mathrm{h}_{1}+\mathrm{h}_{3}=0.1092 \mathrm{~m}<\frac{\mathrm{t}}{2} \tag{О.K.}
\end{equation*}
$$

## Whining verification

- From Table 6.1 - Item 4 for a $\mathrm{W}=0.80 \mathrm{~T}$ we get $\mathrm{Z} / 2=0.1991 \mathrm{~T}$, therefore $\mathrm{Z}=$ 0.199 m.
- We calculate the whining rate (Vow) of Equation 6.46.

$$
\mathrm{V}_{\mathrm{oW}}=\frac{0.0229 \sigma \mathrm{~g}_{\mathrm{c}}}{\mu_{\mathrm{G}}}\left(\frac{\mu_{\mathrm{G}}^{2}}{\sigma \mathrm{~g}_{\mathrm{c}} \rho_{\mathrm{G}} \mathrm{~d}_{\mathrm{o}}} \frac{\rho_{\mathrm{L}}}{\rho_{\mathrm{G}}}\right)^{0.379}\left(\frac{\mathrm{l}}{\mathrm{~d}_{\mathrm{o}}}\right)^{0.293}\left(\frac{2 \mathrm{~A}_{\mathrm{d}} \mathrm{~d}_{\mathrm{o}}}{\sqrt{3} \mathrm{p}^{\prime 3}}\right)^{2.8 /\left(\mathrm{Z} / \mathrm{d}_{\mathrm{o}}\right)^{0.724}}=3.85 \mathrm{~m} / \mathrm{s}
$$

- whining check:

$$
\begin{equation*}
\mathrm{V}_{\mathrm{o}}=5.1 \mathrm{~m} / \mathrm{s}>\mathrm{V}_{\mathrm{ow}} \tag{0.K.}
\end{equation*}
$$

## Liquid Crawl Verification

- To determine the drag (E) we will use Figure 6.17, for which we enter by the abscissa with the value of:

$$
\frac{\mathrm{L}^{\prime}}{\mathrm{G}^{\prime}}\left(\frac{\rho_{\mathrm{G}}}{\rho_{\mathrm{L}}}\right)^{0.5}=0.12
$$

- Up to the curve $\mathrm{V} / \mathrm{V}_{\mathrm{F}}=0.70$, therefore of the figure: $\quad \mathbf{E}=\mathbf{0 . 0 1 5}$
- The drag is so small that it does not significantly modify the hydraulics of the plate.


## Global plate efficiency

- To determine the overall efficiency of the plate (Eo) we will use Figure 6.25, for which we enter by the abscissa with the value of:

$$
\alpha_{A V} \mu_{L}=1.5 \times 10^{-3}
$$

- Therefore of the figure: $\quad$ Eo $=\mathbf{0 . 4 5}$

| TOWER DISTILLATION IPA |  |  |
| :---: | :--- | :---: |
| $\mathrm{T}(\mathrm{m})$ | Diameter of the tower | 0.50 |
| Nr | Number of actual dishes | 34 |
| Nf | Food dish | 32 |
| $\mathrm{H}(\mathrm{m})$ | Tower height | 8.25 |
| $\mathrm{e}(\mathrm{mm})$ | Wall Thickness | 5 |
| $\mathrm{M} \mathrm{(kg)}$ | Mass of cover | 520 |

## ANNEX N ${ }^{\circ} 13$

## IPA Column Cap Condenser

From the following data:

| Current | Inlet | Outlet | L1 | L2 |
| :---: | :---: | :---: | :---: | :---: |


| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 101.00 | 96.45 | 21.85 | 40.00 |
| :--- | :---: | :---: | :---: | :---: |
| Temperature (K) | 374.15 | 369.60 | 295.00 | 313.15 |
| Pressure (kPa) | 255.00 | 250.00 | 300.00 | 300.00 |
| Fraction of vapor | 1.00 | 0.00 | 0.00 | 0.00 |
| Molar flow (kmol / hr) | 12.9484 | 12.9484 | 393.4588 | 393.4588 |
| Components |  |  |  |  |
| Molar flow (kmol / hr) |  |  |  |  |
| H20 | 6.2730 | 4.2730 | 393.4588 | 393.4588 |
| IPA | 0.0000 | 0.0000 | 0.0000 | 0.0000 |
| HYDROGEN | 2.5847 | 2.5847 | 0.0000 | 0.0000 |
| ACETONE |  |  |  |  |
| Molar fractions | 0.3300 | 0.3300 | 1.000 | 1.000 |
| H20 | 0.4704 | 0.4704 | 0.000 | 0.000 |
| IPA | 0.0000 | 0.0000 | 0.000 | 0.000 |
| HYDROGEN | 0.1996 | 0.1996 | 0.000 | 0.000 |
| ACETONE | 593.1232 | 593.1232 | 7088.2002 | 7088.2002 |
| Mass flow (kg / h) |  |  |  |  |

## A. Determination of water vapor flow.

The heat required for the condensation of the inlet flow is determined from the difference in enthalpy between the inlet and outlet conditions, for which the bubble and dew point temperature of the inlet stream was defined $\left(\mathrm{T}=101^{\circ} \mathrm{C}\right.$ and bubble $\left.\mathrm{T}=96.45^{\circ} \mathrm{C}\right)$.

$$
Q_{p}=474000 \frac{\mathrm{~kJ}}{\mathrm{~h}}
$$

The flow rate of the cooling water (295 K):

$$
\dot{m}_{\text {water vapor }}=\frac{474000}{66.8717}=7088.20024 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

## B. Determination of LMTD.

$$
L M T D=\frac{(374.15 K-313.15 K)-(369.60 K-295.00 K)}{\ln \left(\frac{(374.15 K-313.15 K)}{(369.60 K-295.00 K)}\right)}=67.5721 \mathrm{~K}
$$

## C. Determination of the overall coefficient.

- Coefficient of heat transfer per armor - ho.

Outer tube diameter: $\quad D_{o}=0.0191 \mathrm{~m}^{2}$
Separation between Pitch and Do: $\quad C^{\prime}=0.0064 \mathrm{~m}^{2}$ Armor Diameter: $\quad D_{c}=0.3874 m(15.25$ pulg. $)$

Space between baffles: $B=0.0775 m$
Armored area: $\quad A s=\frac{0.0064 \times 0.3874 \times 0.0775}{0.0254}=0.0076 \mathrm{~m}^{2}$
Water mass flow: $\quad G_{t}=\frac{7088.2002 \frac{\mathrm{~kg}}{\mathrm{~h} . \mathrm{m2}}}{0.0076 \mathrm{~m}^{2} \times 3600}=260.405 \frac{\mathrm{~kg}}{\mathrm{s.m}^{2}}$
To $T$ average $=30.93^{\circ} \mathrm{C}$, the properties of the cooling water are:

$$
\begin{aligned}
\mu & =0.00076 \mathrm{~Pa} . \mathrm{s} \\
C p & =4.2267 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}} \\
k & =0.6217 \frac{\mathrm{~W}}{\mathrm{~m} \cdot \mathrm{~K}}
\end{aligned}
$$

The equivalent diameter of the armor:

$$
D e=4 x\left(\frac{0.8661 x\left(\left(\left(\frac{0.0254}{0.0254}\right)\right)^{2}-\frac{\pi}{4} x\left(\frac{0.0191}{0.0254}\right)^{2}\right)}{\pi x\left(\frac{0.0191}{0.0254}\right)}\right) x 0.0254=0.01815 \mathrm{~m}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{260.405 \times 0.01815 \mathrm{~m}}{0.00076 \text { Pa. } . \mathrm{s}}=6299.13
$$

From Fig. 28 of the book "Heat Transfer Processes" by Donald Kern, Factor Jh is extracted for the given Reynolds number:


So:

$$
h o=J h x \frac{k}{D e} \times\left(\frac{C \mu}{k}\right)^{1 / 3}=6785.5673 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

$$
\text { Internal area of a tube: } \quad A_{t^{\prime}}=\frac{\pi x D i^{2}}{4}=0.00019 \mathrm{~m}^{2}
$$

$$
\begin{gathered}
\text { Pipe area: } \quad A_{t}=\frac{N t \times A_{t}}{\text { Number of steps }}=\frac{122 \times 0.00019}{2}=0.01182 \mathrm{~m}^{2} \\
\text { Mixing Flux: } \quad G_{t}=\frac{593.1232 \mathrm{~kg} / \mathrm{h}}{0.01182 \mathrm{~m}^{2}}=13.94 \frac{\mathrm{~kg}}{\mathrm{s.m}}
\end{gathered}
$$

To $T_{m}=96.45^{\circ} \mathrm{C}$, the properties of the mixture are:

$$
\begin{gathered}
\mu=0.000008 \mathrm{Pa.s} \\
C p=1.7324 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}}
\end{gathered}
$$

$$
k=0.0370 \frac{W}{m \cdot K}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{13.94 \times 0.0157 \mathrm{~m}}{0.000008 \text { Pa.s }}=25933.93
$$

From Fig. 24 of the book "Heat Transfer Processes" by Donald Kern, Factor Jh is extracted for the given Reynolds number:


$$
J h=160.00
$$

Así:

$$
h i=\operatorname{Jh} \times \frac{k}{D e} x\left(\frac{C \mu}{k}\right)^{1 / 3}=90.3933 \frac{W}{m^{2} . K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{6785.5673 \times 90.3933}{6785.5673+90.3933}=89.2050 \frac{W}{m^{2} \cdot K}
$$

Area required for heating:

$$
A 1=\frac{\frac{474000}{67.5721 \times 3.6}}{89.2050}=32.9117 \mathrm{~m}^{2}
$$

## D. Checking of maximum heat flow.

- Total clean area Ac:

$$
A c=A 1=32.9117 \mathrm{~m}^{2}
$$

- Total design coefficient:

Total area $=$ Nt $\times L T \times R T=122 \times 5 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m}=$ $36.4977 \mathrm{~m}^{2}$ :

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Tube surface ratio per linear foot (BWG 16)

$$
U D=\frac{\frac{474000}{67.5721 \times 3.6}}{36.4977}=53.3879 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot K}
$$

## ANNEX N ${ }^{\circ} 14$

## IPA reflux pumps

From the following data obtained from the material balance.

| Current | IPA recycle |
| :--- | :--- |


| Temperature ( ${ }^{\circ}$ C) | 96.450 | 96.450 |
| :--- | :---: | :---: |
| Pressure (kPa) | 230.00 | 250.00 |
| Fraction of vapor | 0 | 0 |
| Molar flow (kmol / <br> hr) | 2.2134 | 2.2134 |
| Components | 0. |  |
| Molar flow (kmol / <br> hr) | 0.442 | 0.442 |
| HYDROGEN | 1.041 | 1.041 |
| ACETONE | 0.7304 | 0.7304 |
| IPA | 101.39 | 101.39 |
| H20 | 710.5000 | 710.5000 |
| Mass flow (kg / h) | 0.00004 | 0.00004 |
| Density (kg / m3) |  |  |
| Flow rate (m3 / s) |  |  |

Performing energy balance at pump inlet and outlet:

$$
\begin{gathered}
\mathrm{E}_{1}+\mathrm{H}_{\mathrm{B}}=\mathrm{E}_{2} \\
\frac{\mathrm{P}_{1}}{\gamma}+\mathrm{Z}_{1}+\frac{\overline{\mathrm{V}}_{1}}{2 \mathrm{~g}}+\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}+\mathrm{Z}_{2}+\frac{\overline{\mathrm{V}}_{2}}{2 \mathrm{~g}} \\
\mathrm{H}_{\mathrm{B}}=\frac{\mathrm{P}_{2}}{\gamma}-\frac{\mathrm{P}_{1}}{\gamma}=\frac{300.2 \mathrm{kPa}}{9.81 \mathrm{~N} \times 777.6 \frac{\mathrm{~kg}}{\mathrm{~m}^{3}}}=39.354 \mathrm{~m}
\end{gathered}
$$

For the design, the data of the impeller diameter, roughness, rotational speed.

| Design data |  |
| :--- | :---: |
| Impeller Diameter Da (m) | 0.371 |


| Pump head H (m) | 2.8694 |
| :--- | :---: |
| Rotational speed n (1 / sec) | 1.4 |
| Power of pump P (W) | 0.7927761 |
| Discharge flow Q (m3 / s) | 0.00004 |
| Viscosity (Kg / m.s) | 0.001 |
| Density (Kg / m3) | 710.5000 |
| Roughness $\square$ (m) | 0.0001 |
| Coefficient of capacity Cq | 0.0006 |
| Coefficient of head Ch | 104.343 |
| Power factor CP | 0.0579 |

## ANNEX N ${ }^{\circ} 15$

## Column bottom reboiler IPA

From the following data:

| Current | B | V | $\mathrm{V}+\mathrm{B}$ | V 1 | V 2 |
| :--- | :---: | :---: | :---: | :---: | :---: |
| Temperature ( ${ }^{\circ} \mathrm{C}$ ) | 129.94 | 129.94 | 127.73 | 165.00 | 165.00 |
| Temperature (K) | 403.09 | 403.09 | 400.88 | 438.15 | 438.15 |
| Pressure (bar) | 270.00 | 270.00 | 275 | 7.00 | 7.00 |
| Fraction of vapor | 0.00 | 1.00 | 0.00 | 1.00 | 0.00 |
| Molar flow (kmol / hr) | 22.4849 | 22.4849 | 35.4332 | 166.7445 | 166.7445 |
| Components |  |  |  |  |  |
| Molar flow (kmol / hr) |  |  |  |  |  |
| H20 | 22.4826 | 12.9483 | 35.4310 | 166.7445 | 166.7445 |
| IPA | 0.0022 | 0.0000 | 0.0022 | 0.0000 | 0.0000 |
| HYDROGEN | 0.0000 | 0.0000 | 0.0000 | 0.0000 | 0.0000 |
| ACETONE | 0.0000 | 0.0000 | 0.0000 | 0.0000 | 0.0000 |
| Molar fractions |  |  |  |  |  |
| H20 | 0.9999 | 0.9999 | 0.9999 | 1.000 | 1.000 |
| IPA | 0.0001 | 0.0001 | 0.0001 | 0.000 | 0.000 |
| HYDROGEN | 0.0000 | 0.0000 | 0.0000 | 0.000 | 0.000 |
| ACETONE | 0.0000 | 0.0000 | 0.0000 | 0.000 | 0.000 |
| Mass flow (kg / h) | 405.1600 | 233.3419 | 638.5019 | 3003.9186 | 3001.4008 |

## E. Heat balance.

For design considerations for a vaporizer, only $80 \%$ of the inlet flow to the schematic can be vaporized:

$$
\text { Vapor } \text { Ratio }=\frac{12.9484}{35.4333}=0.3654
$$

The outline of the team is as follows:


For the determination of the overall transfer coefficient, we have the following calculation sequence:

## F. Determination of water vapor flow.

The heat required for the vaporization of the inlet flow is:

$$
Q_{p}=6227000 \frac{\mathrm{~kJ}}{\mathrm{~h}}
$$

The required steam flow rate ( $165^{\circ} \mathrm{C}$ and 700 kPa ):

$$
\dot{m}_{\text {water vapor }}=\frac{6227000}{2072.959}=3003.9186 \frac{\mathrm{~kg}}{\mathrm{~h}}
$$

## G. Determination of LMTD.

Since the heating medium (water vapor) has a constant temperature profile, it is feasible to calculate the LMTD as follows:

$$
L M T D=\frac{(438.15 K-403.09 K)-(438.15 K-400.88 K)}{\ln \left(\frac{(438.15 K-403.09 K)}{(438.15 K-400.88 K)}\right)}=36.1537 K
$$

## H. Determination of the overall coefficient.

- Coefficient of heat transfer per armor - ho.

Outer tube diameter: $\quad D_{o}=0.0191 \mathrm{~m}^{2}$

Separation between Pitch and Do: $\quad C^{\prime}=0.0064 \mathrm{~m}^{2}$

Armor Diameter: $\quad D_{c}=0.3366 m(15.25$ pulg.)

Space between baffles: $\quad B=0.0673 m$

Armored area: $\quad$ As $=\frac{0.0064 \times 0.3366 \times 0.0673}{0.0254}=0.0057 \mathrm{~m}^{2}$

Mixing Flux: $\quad G_{t}=\frac{7791.7273 \frac{\mathrm{~kg}}{\mathrm{~h} . \mathrm{m2}}}{0.0057 \mathrm{~m}^{2} \times 3600}=379.189 \frac{\mathrm{~kg}}{\mathrm{~s} . \mathrm{m}^{2}}$

To $T m=129.94^{\circ} \mathrm{C}$, the properties of the fluid are:

$$
\mu=0.00021 \text { Pa.s }
$$

$$
C p=4.2546 \frac{\mathrm{~kJ}}{\mathrm{~kg} \cdot \mathrm{~K}}
$$

$$
k=0.6878 \frac{W}{m \cdot K}
$$

The equivalent diameter of the armor:

$$
D e=4 x\left(\frac{0.8661 x\left(\left(\left(\frac{0.0254}{0.0254}\right)\right)^{2}-\frac{\pi}{4} x\left(\frac{0.0191}{0.0254}\right)^{2}\right)}{\pi x\left(\frac{0.0191}{0.0254}\right)}\right) x 0.0254=0.01815 \mathrm{~m}
$$

The Reynolds number is:

$$
R e=\frac{G_{t} \times D t}{\mu}=\frac{286.252 \times 0.01815 \mathrm{~m}}{0.00021 \text { Pa. } \mathrm{s}}=32586.36
$$

From Fig. 24 of the book "Heat Transfer Processes" by Donald Kern, Factor Jh is extracted for the given Reynolds number:


$$
J h=100.00
$$

So:

$$
\text { Ho }=\operatorname{Jh} x \frac{k}{D e} x\left(\frac{C \mu}{k}\right)^{1 / 3}=4144.0546 \frac{W}{m^{2} \cdot K}
$$

- Determination of the coefficient of heat transfer by tubes - hi.

For purposes of calculation, the hi recommended by Donald Kern for steam flow:

$$
h_{i}=8000 \frac{W}{m^{2} \cdot K}
$$

For a clean overall coefficient (U1):

$$
U 1=\frac{8000 \times 4144.0546}{8000+4144.0546}=2729.93 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot \mathrm{~K}}
$$

Area required for heating:

$$
A 1=\frac{\frac{6227000}{36.1537 \times 3.6}}{2729.93}=17.5256 \mathrm{~m}^{2}
$$

## I. Checking the maximum flow of heat.

- Total clean area Ac:

$$
A c=A 1=17.5256 \mathrm{~m}^{2}
$$

- Total design coefficient:

Total area $=N t \times L T \times R T=86 \times 5 \times 0.1963 \frac{f t^{2}}{f t} \times 0.3048 \mathrm{~m}=$ $25.7279 \mathrm{~m}^{2}$ :

Where:
o Nt: Number of tubes in the shell
o LT: Length of tubes
o RT: Tube surface ratio per linear foot (BWG 16)

$$
U=\frac{\frac{6227000}{3.6 \times 36.1537}}{25.7279}=1859.5989 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot K}
$$

## ANNEX N ${ }^{\circ} 16$

## Storage Tanks for IPA and Acetone

- With the data of capacity, service factor and the balance of the process we determine the program of production:

| Nominal Capacity (MTM / <br> Year) | 21.5 |
| :---: | :---: |
| Service factor (\%) | $93.2 \%$ |
| TM IPA / TM Acetone | 1.34 |


| Production program | MTM/Año | TM/DO |
| :---: | :---: | :---: |
| Production Acetone | 20.0 | 58.8 |
| IPA Requirement | 26.7 | 78.7 |

- Storage capacity is required for 14 days of raw material and finished product, for which:

| Density Acetone (TM / m3) | 0.791 |
| :---: | :---: |
| IPA Density (TM / m3) | 0.786 |$\quad$| Inventory (days) |  |
| :---: | :---: |
| Acetone | 14 |
| IPA | 14 |


| Storage | TM | m3 |
| :---: | :---: | :---: |
| Acetone | 824 | 1041.1 |
| IPA | 1101 | 1401.0 |

- The tanks will have the following dimensions and with this we will determine the number of tanks that are required:

| Dimensions of the tank (m) |  | Tank volume (m3) |  |
| :---: | :---: | :---: | :---: |
| D (diameter) | 10 | Total V | 392.7 |
| H (height) | 5 | V filling (80\%) | 314.2 |


| Number of tanks |  |  |
| :---: | :---: | :---: |
| Acetone | 3.31 | 4 |
| IPA | 4.46 | 5 |

# DECLARATION OF CONFIDENTIALITY 

Eng. Rafael Chero Rivas, Eng. Víctor León Choy,

Professors responsible for the course Design of Plants PI-525 / A

At the request of the students Alex Renzo Condori Llacta, Fabrizio Alexander Flores Estrada, Kevin Andrei Flores Gil, Miguel Eduardo Soto Moreno, authors of the final report titled "Design of an acetone production plant via catalytic dehydrogenation of isopropyl alcohol".

## DECLARE

The confidentiality of the report in question with the effects that it derives from the terms of the corresponding instruction, which regulates the management of undergraduate studies in the Faculty of Chemical and Textile Engineering in respect of final reports, submitted Confidentiality.

Lima, December 282015.

## Alex Renzo Condori Llacta

20114099K

> Fabrizio Alexander Flores Estrada 20114049C

Kevin Andrei Flores Gil<br>20112140C

